

# Lead-Lag Compensation

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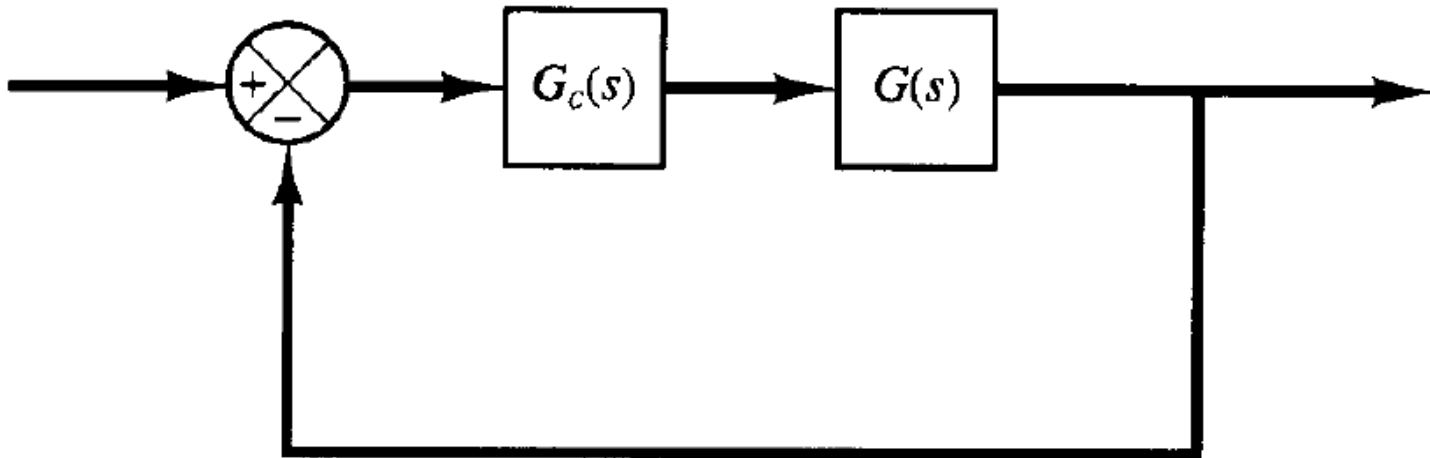
# “lead” and “lag”

- Adding a pole to the system changes the phase by **-90 deg** and adding a zero changes the phase by **+90 deg**.
- Generally the purpose of the Lead-Lag compensator is to create a controller which has an overall magnitude of approximately 1.
- It is used for phase compensation rather than magnitude.

# Lead Compensator

- Assume the following lead compensator is :

$$G_c(s) = K_c \alpha \frac{Ts + 1}{\alpha Ts + 1} = K_c \frac{s + \frac{1}{T}}{s + \frac{1}{\alpha T}} \quad 0 < \alpha < 1$$



- where  $G(s) = \frac{4}{s(s+2)}$
- It is **desired to design** a compensator for the system so that the static velocity error constant  $K_v$  is  $20 \text{ sec}^{-1}$ , the phase margin is at least  $50^\circ$ , and the gain margin is at least 10 dB.
- Define  $K_c \alpha = K$
- Then  $G_c(s) = K \frac{Ts + 1}{\alpha Ts + 1}$

- **The open-loop transfer function of the system is**

$$G_c(s)G(s) = K \frac{Ts + 1}{\alpha Ts + 1} G(s) = \frac{Ts + 1}{\alpha Ts + 1} G_1(s)$$

- **and**  $G_1(s) = KG(s) = \frac{4K}{s(s + 2)}$

- **therefore,**  $K_v = \lim_{s \rightarrow 0} s G_c(s)G(s) = \lim_{s \rightarrow 0} s \frac{Ts + 1}{\alpha Ts + 1} G_1(s)$

$$= \lim_{s \rightarrow 0} \frac{s4K}{s(s + 2)} = 2K = 20$$

- **Next plot the bode diagram of**

$$G_1(j\omega) = KG(j\omega) = \frac{40}{j\omega(j\omega + 2)} = \frac{20}{j\omega(0.5j\omega + 1)}$$

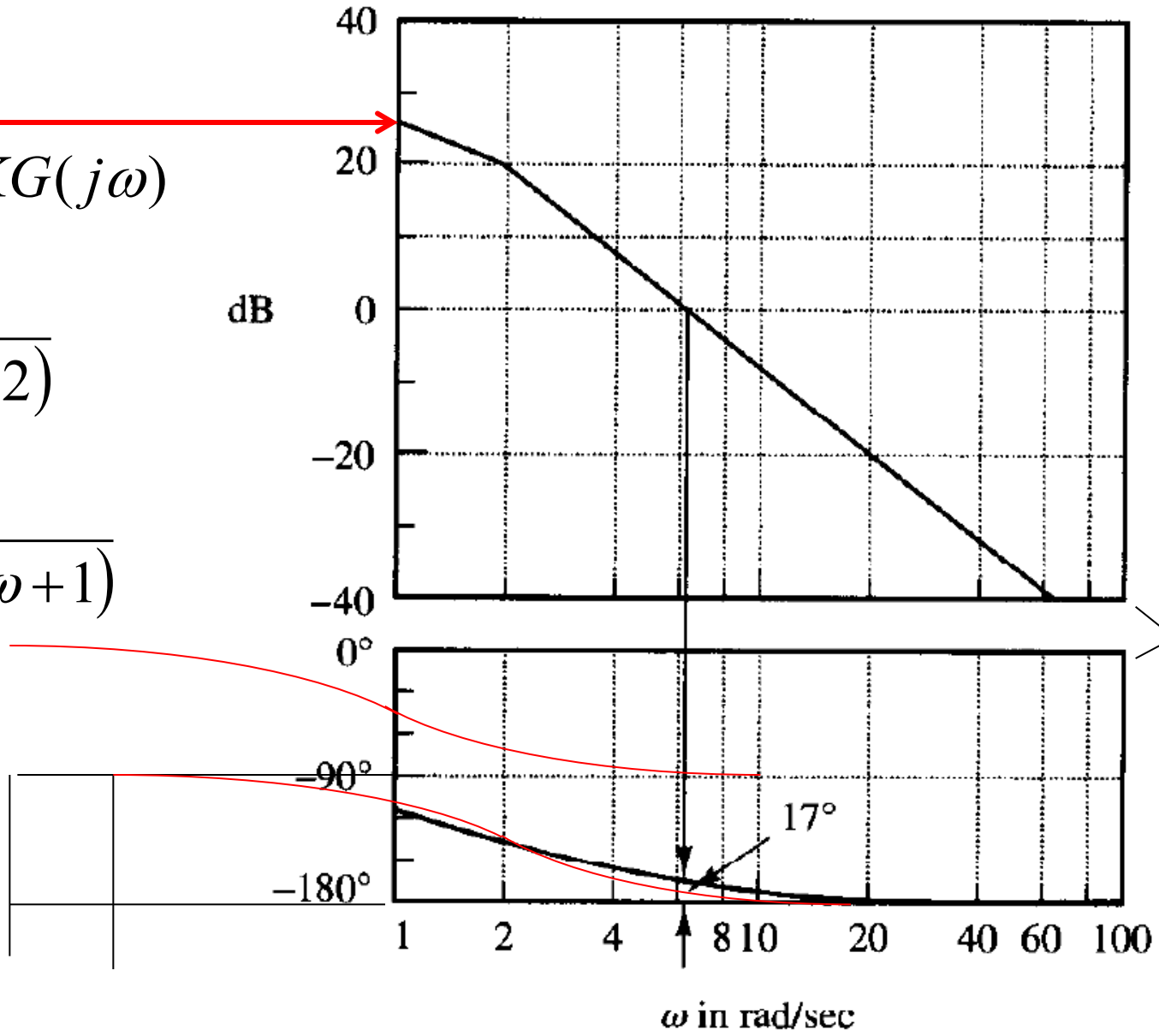
- **The transfer function has 3 components:**
  - **A constant of 20**—which is equal to **26.0 dB**. The phase is constant at **0 degrees**.
  - **A pole at s=0**—break at **1rad/sec**, a pure integral with slope **-20dB/decade**
  - **The phase is 0 degrees up to 1/10 the break frequency (0.1 rad/sec)** then drops linearly down to **-90 degrees at 10 times the break frequency (10 rad/sec)**.

- A pole at  $s=-2$ —break at 2rad/sec, with slope of  $(-20)+(-20)$ dB/decade
- The phase is 0 degrees up to **1/10** the break frequency (0.2 rad/sec) then drops linearly down to -180 degrees at **10 times** the break frequency (20 rad/sec).

$$G_1(j\omega) = KG(j\omega)$$

$$= \frac{40}{j\omega(j\omega + 2)}$$

$$= \frac{20}{j\omega(0.5j\omega + 1)}$$



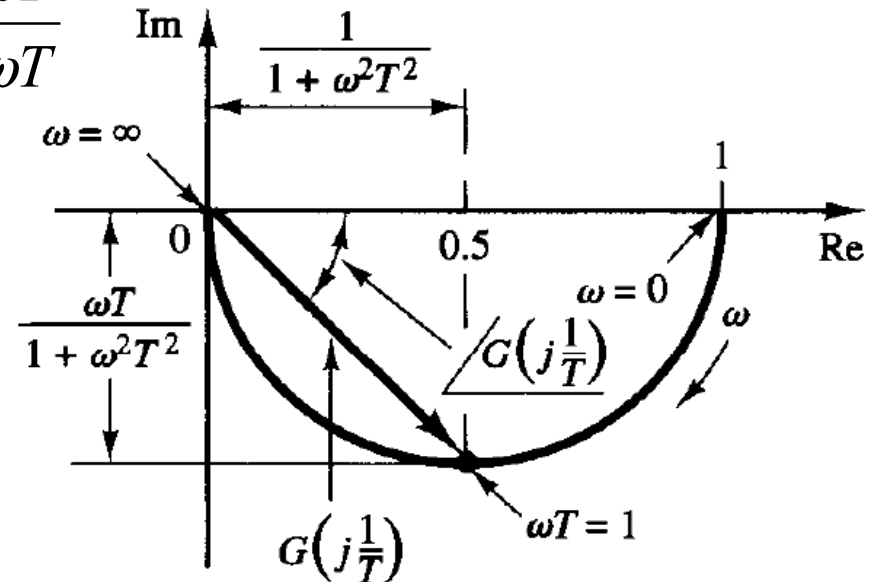


- The phase margin of the system is  $17^\circ$  and the gain margin is  $+\infty$ dB
- Phase margin of  $17^\circ$  implies that the system is **quite oscillatory**.
- The specification calls for a phase margin of at least  $50^\circ$ .
- A **lead compensator** is needed for this purpose.

# Polar Plot

- Consider the Nyquist plots of the First order factor  $(1 + s)^{-1} \rightarrow (1 + j\omega)^{-1}$ .
- The sinusoidal transfer function

$$\begin{aligned}
 G(j\omega) &= \frac{1}{1 + j\omega T} = \frac{1}{1 + j\omega T} \frac{1 - j\omega T}{1 - j\omega T} \\
 &= \frac{1}{1 + \omega^2 T^2} + j \frac{-\omega T}{1 + \omega^2 T^2} \\
 &= \frac{1}{\sqrt{1 + \omega^2 T^2}} (-\arctan(\omega T))
 \end{aligned}$$

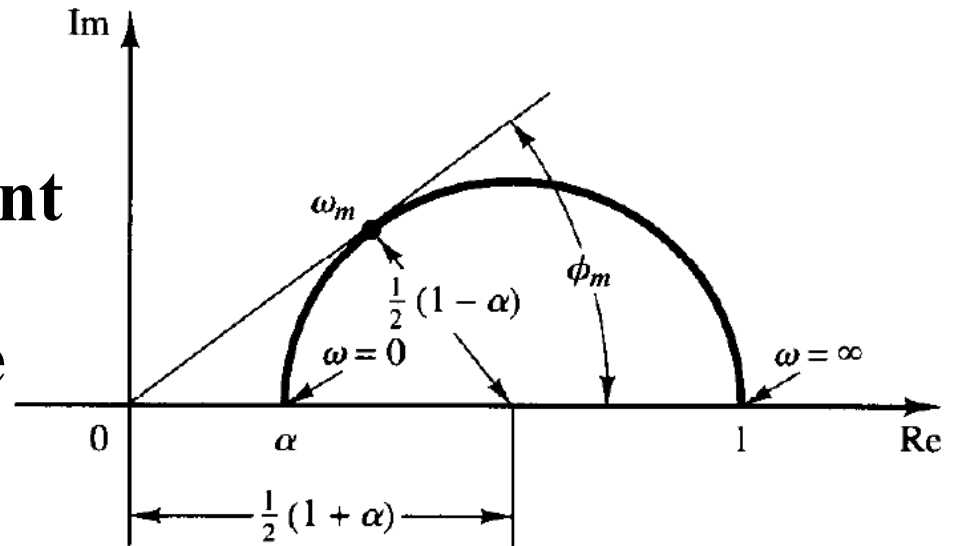


# similarly

- Polar plot of

$$G_c(s) = K_c \alpha \frac{Ts + 1}{\alpha Ts + 1} = K_c \frac{s + \frac{1}{T}}{s + \frac{1}{\alpha T}} \quad 0 < \alpha < 1$$

- With  $K_c = 1$ , for a given value  $\alpha$ , the angle between the positive real axis and the tangent line drawn from the origin to the semicircle gives the **maximum phase lead angle**.



- **Phase angle at  $\omega = \omega_m$  is  $\varphi_m$ , where**

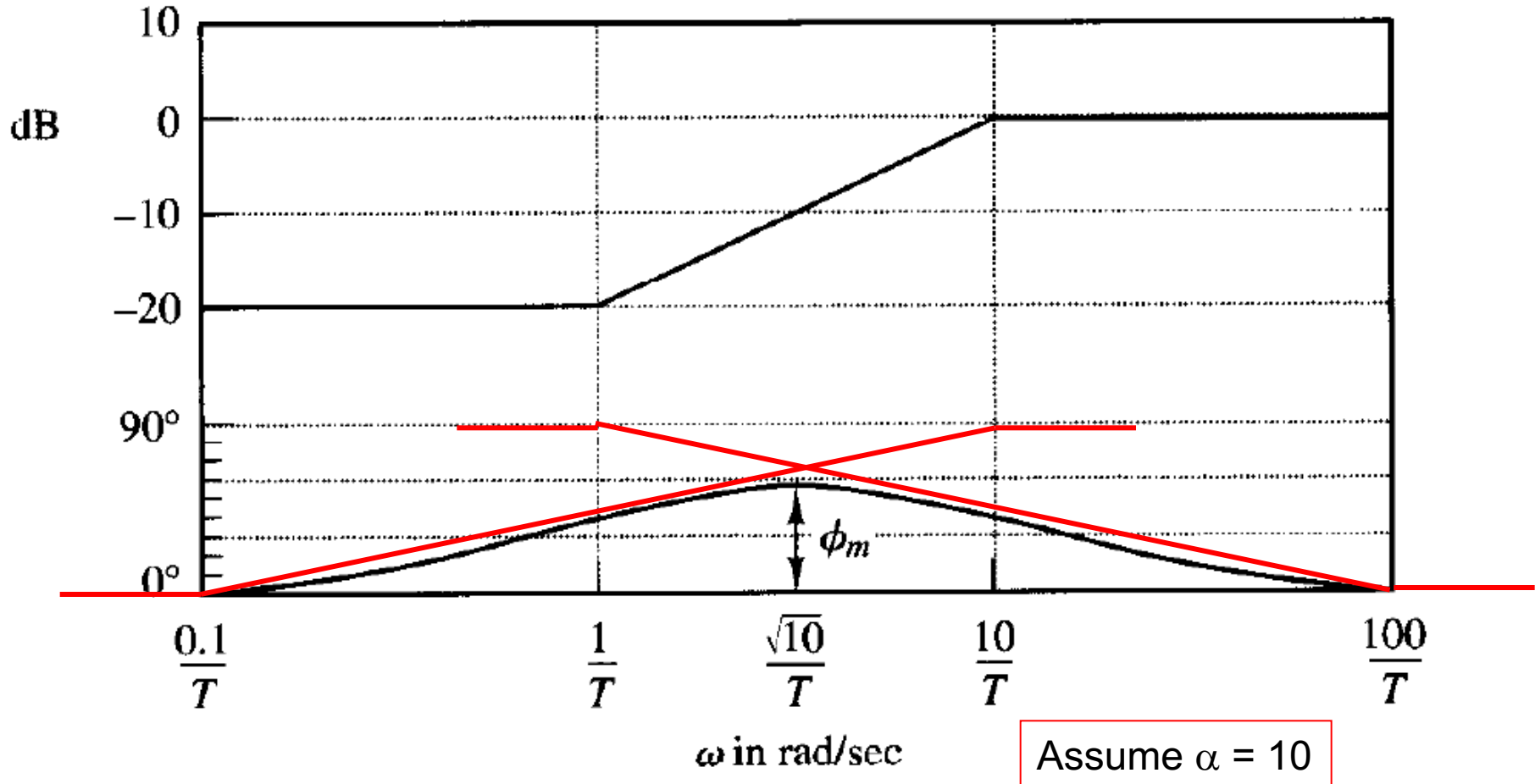
$$\sin \varphi_m = \frac{\frac{1-\alpha}{2}}{\frac{1+\alpha}{2}} = \frac{1-\alpha}{1+\alpha}$$

- **and  $\omega_m$  is the geometric mean of two corner frequencies,**

$$\log \omega_m = \frac{1}{2} \left( \log \frac{1}{T} + \log \frac{1}{\alpha T} \right)$$

$$\omega_m = \frac{1}{\sqrt{\alpha T}}$$

# Bode Diagram of Lead Compensator



- To have a phase margin of at least  $50^\circ$ , phase lead of  $33^\circ$  is needed
- Notice that the addition of a lead compensator, will cause the **gain crossover frequency shift** to the right.
- We **must offset** the increased phase lag of  $G_1(j\omega)$  due to this increase.
- Assume the  $\varphi_m = 38^\circ$ , therefore  $5^\circ$  is used for the offset

- **Since**  $\sin \varphi_m = \frac{1 - \alpha}{1 + \alpha} = \sin 38^\circ$

- **Therefore,  $\alpha = 0.24$ .**

- **Next**

$$|G_c(j\omega)| = \left| \frac{1 + j\omega T}{1 + j\omega\alpha T} \right|_{\omega = \frac{1}{\sqrt{\alpha}T}} = \left| \frac{1 + j \frac{1}{\sqrt{\alpha}}}{1 + j\alpha \frac{1}{\sqrt{\alpha}}} \right| = \frac{1}{\sqrt{\alpha}} = 6.2dB$$

- **Let  $|G_1(j\omega)| = -6.2dB$  , therefore  $\omega_c = 9 \text{ rad/sec}$ .**

**crossover**

- **Therefore,**  $\frac{1}{T} = \sqrt{\alpha}\omega_c = 4.41$

$$\frac{1}{\alpha T} = \frac{\omega_c}{\sqrt{\alpha}} = 18.4$$

- **The lead compensator thus determined is**

$$G_c = K_c \frac{s + 4.41}{s + 18.4} = K_c \alpha \frac{0.227s + 1}{0.054s + 1}$$

- **where**  $K_c = \frac{K}{\alpha} = \frac{10}{0.24} = 41.7$



- **Thus, the transfer function of the compensator is**

$$G_c = 41.7 \frac{s + 4.41}{s + 18.4} = 10 \frac{0.227s + 1}{0.054s + 1}$$

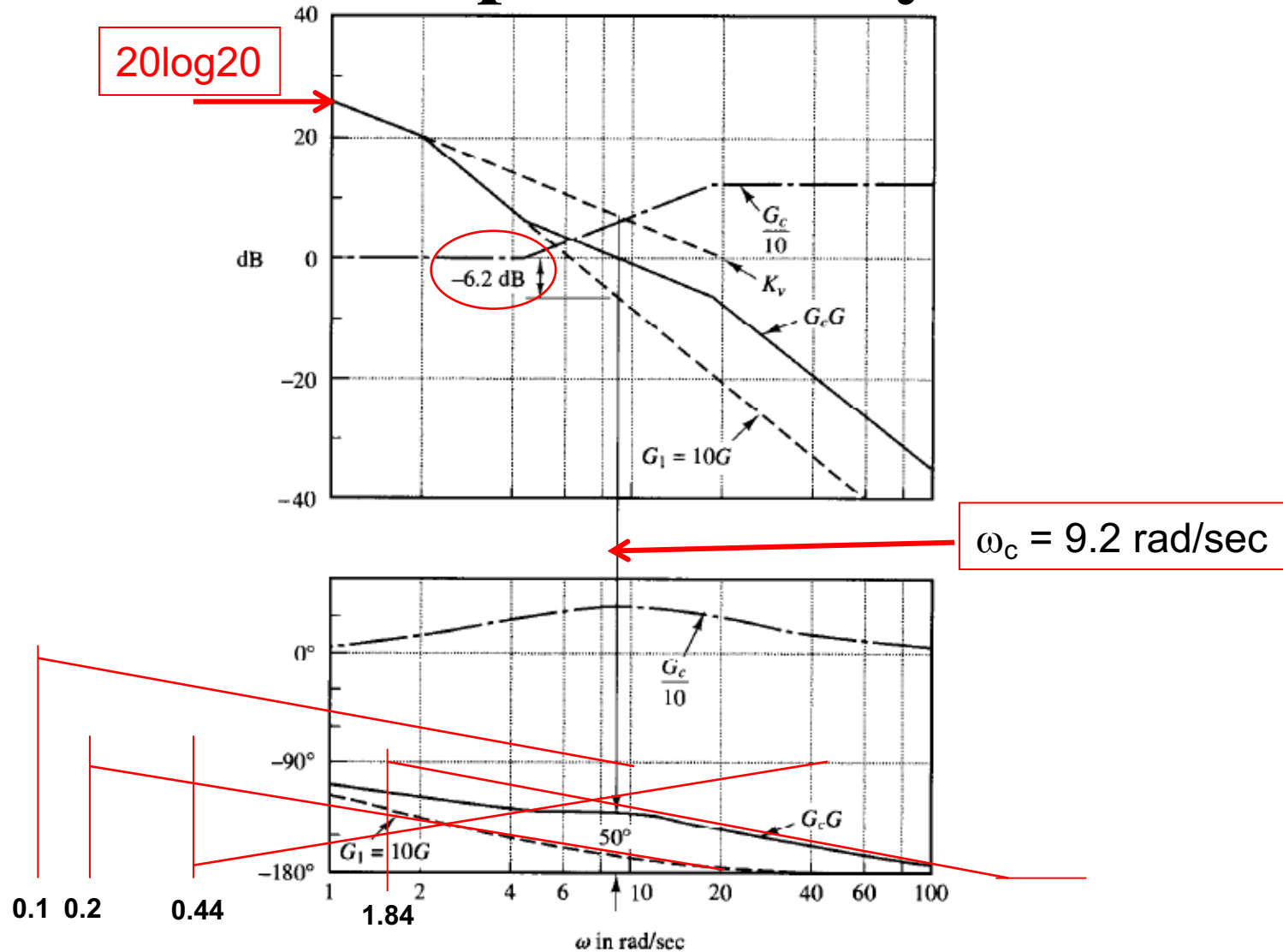
- **and**

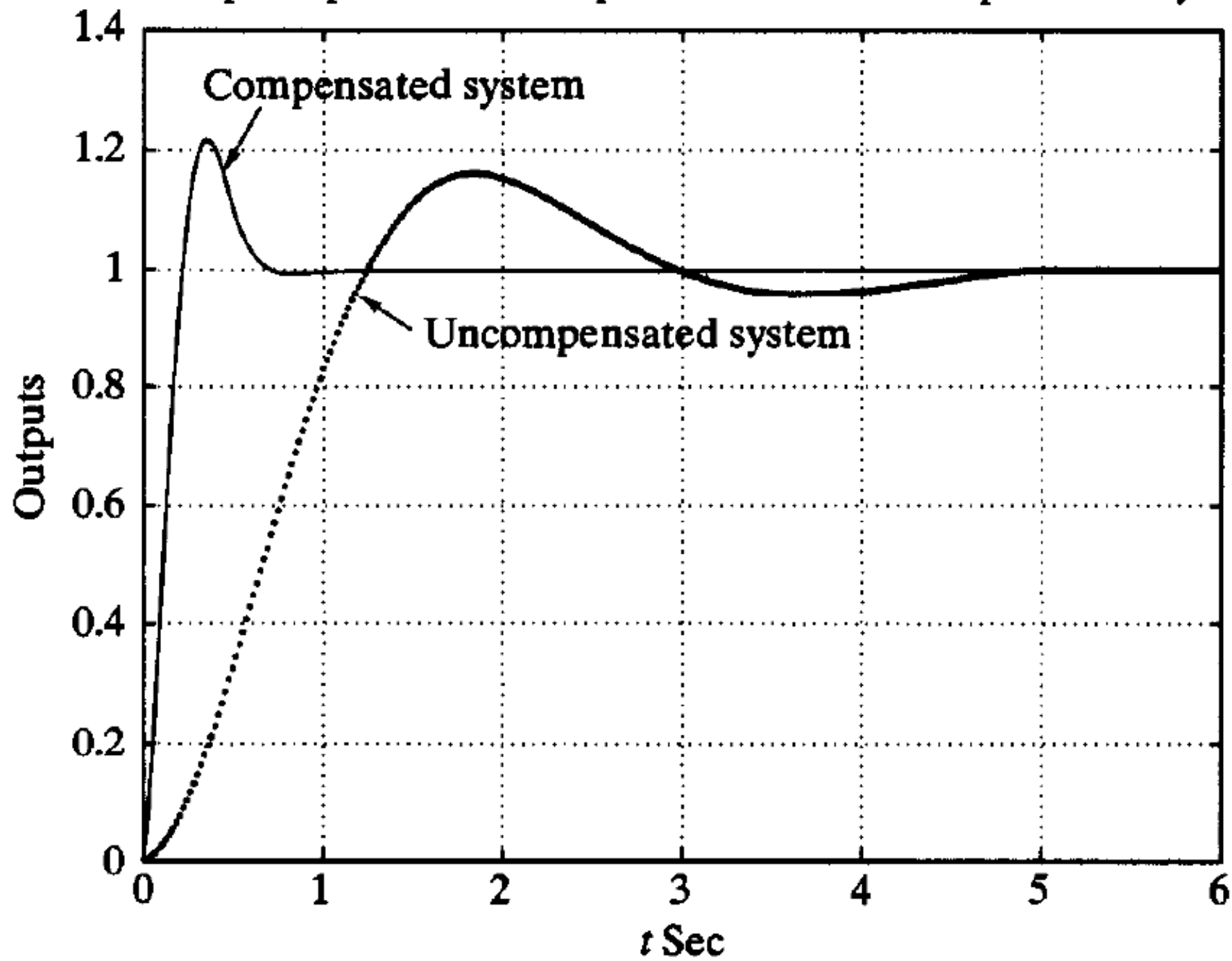
$$\frac{G_c(s)}{K} G_1(s) = \frac{G_c(s)}{10} 10G(s) = G_c(s)G(s)$$

$$G_c(s)G(s) = 41.7 \frac{(s + 4.41)}{(s + 18.4)} \frac{4}{s(s + 2)}$$

$$G_c(s)G(s) = 10 \frac{(.227s + 1)}{(0.054s + 1)} \frac{2}{s(0.5s + 1)}$$

# Bode Diagram of the Compensated System





# Steady state error of cascade compensator

$$G_c(s) = \frac{1 + \alpha Ts}{1 + Ts} \approx 1 + \alpha Ts$$

$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$

$$\frac{C(s)}{R(s)} = \frac{G_c G}{1 + G_c G}$$

$$= \frac{\omega_n^2 + \alpha T \omega_n^2 s}{s^2 + (2\xi\omega_n + \alpha T \omega_n^2 T s) + \omega_n^2}$$

$$2\zeta\omega_n + \alpha T\omega_n^2 = 2\zeta_{eq}\omega_n$$

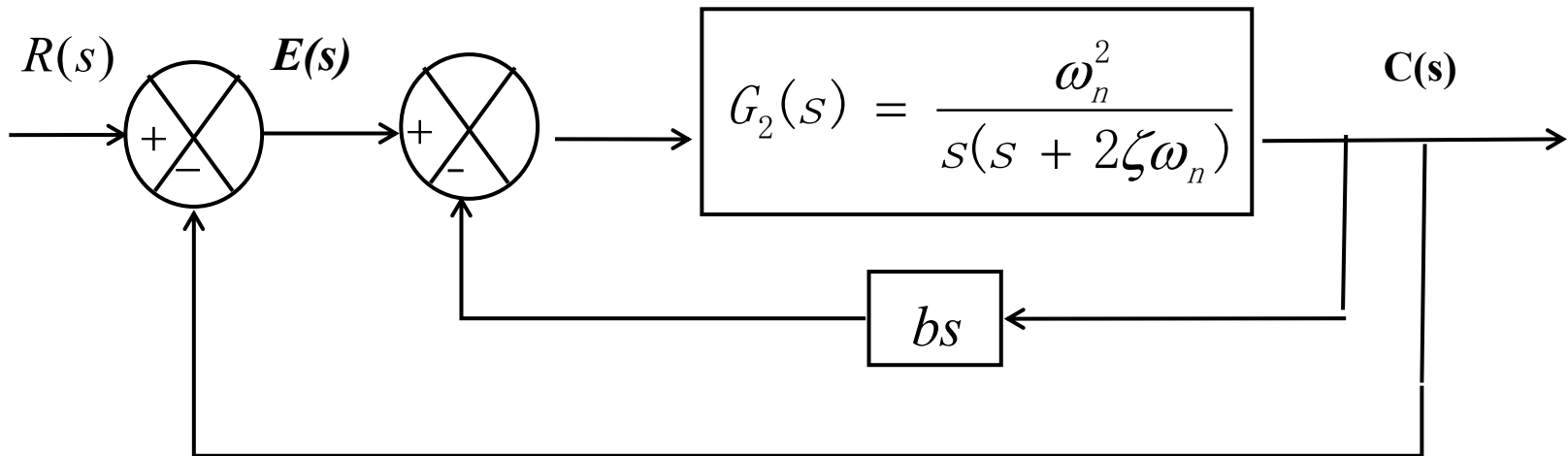
$$\zeta_{eq} = \zeta + \frac{\alpha T\omega_n}{2}$$

$R(s) = 1 / s^2$ , unit ramp input

$$E(s) = \frac{1}{s} \frac{(s + 2\zeta\omega_n)(1 + Ts)}{s(s + 2\zeta\omega_n)(1 + Ts) + \omega_n^2(1 + \alpha Ts)}$$

$$e_{ss(\text{ramp input})} = \lim_{s \rightarrow 0} sE(s) = \frac{2\zeta}{\omega_n}$$

# Minor-loop Feedback Compensation



$$G(s) = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$

$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + (2\xi\omega_n + \omega_n^2 bs) + \omega_n^2}$$

$$2\xi\omega_n + \omega_n^2 b = 2\xi_{eq}\omega_n$$

$$\xi_{eq} = \xi + \frac{\omega_n b}{2}$$

$$R(s) = 1 / s^2, \quad \text{unit ramp input}$$

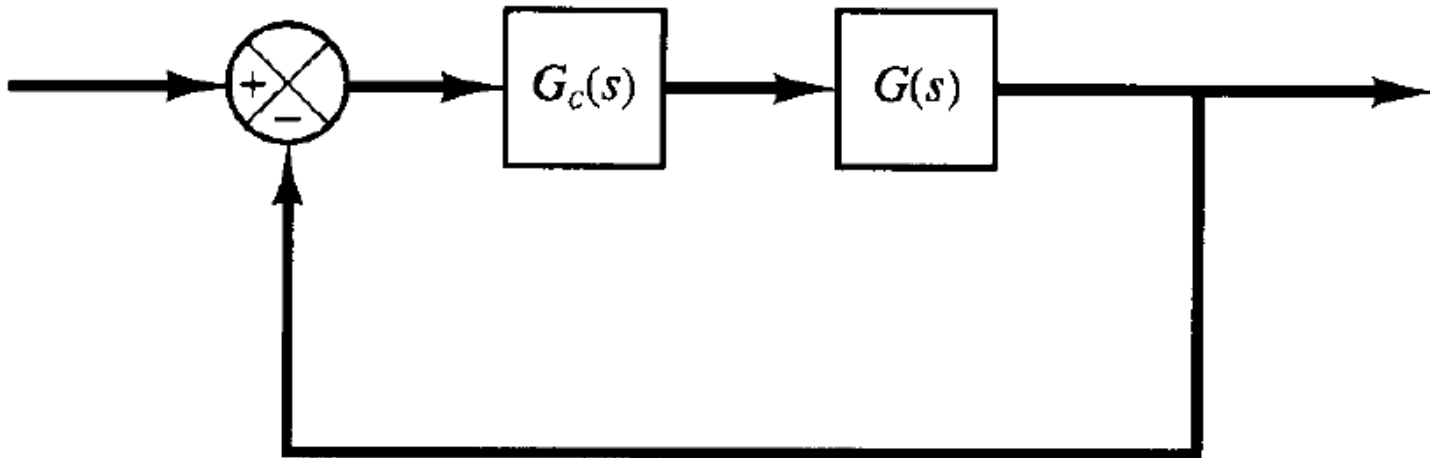
$$E(s) = \frac{1}{s} \left( \frac{s + 2\xi\omega_n + b\omega_n^2}{s(s + 2\xi\omega_n + b\omega_n^2) + \omega_n^2} \right)$$

$$e_{ss(\text{ramp input})} = \lim_{s \rightarrow 0} sE(s) = \frac{2\xi\omega_n + b\omega_n^2}{\omega_n^2} = \frac{2\xi}{\omega_n} + b$$

# Lag Compensator

- Assume the following lag compensator is :

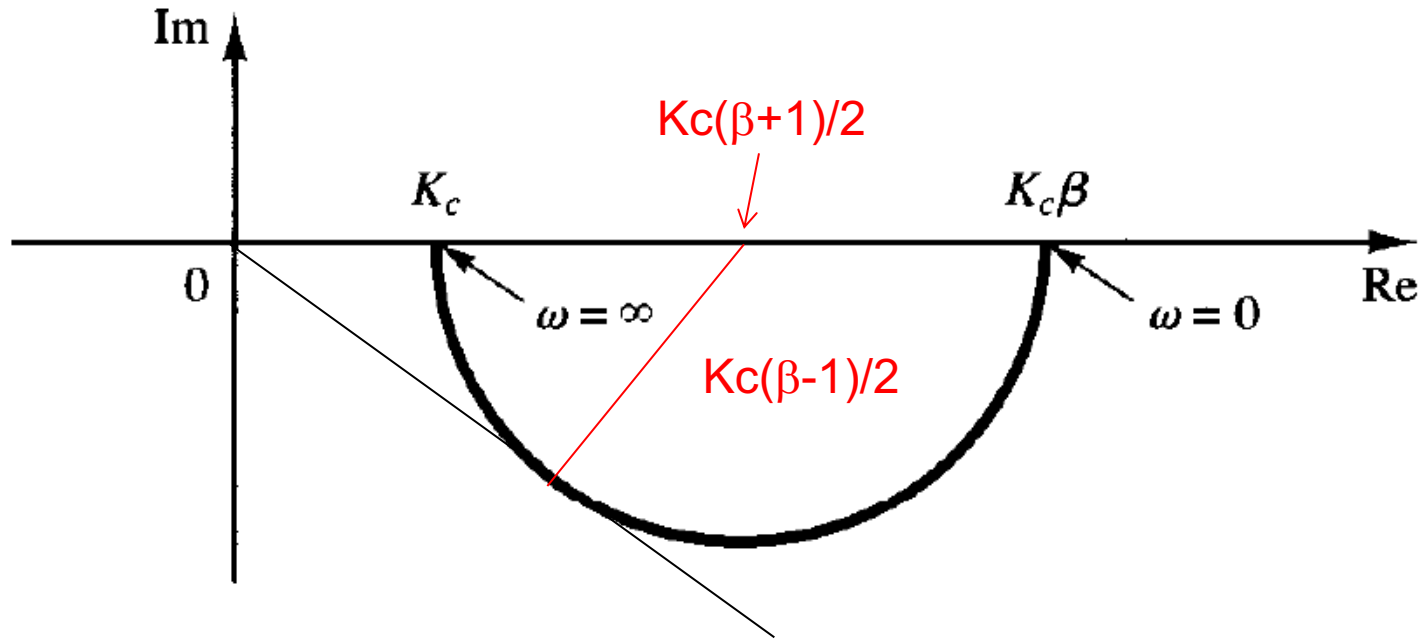
$$G_c(s) = K_c \beta \frac{Ts + 1}{\beta Ts + 1} = K_c \frac{s + \frac{1}{T}}{s + \frac{1}{\beta T}} \quad \beta > 1$$



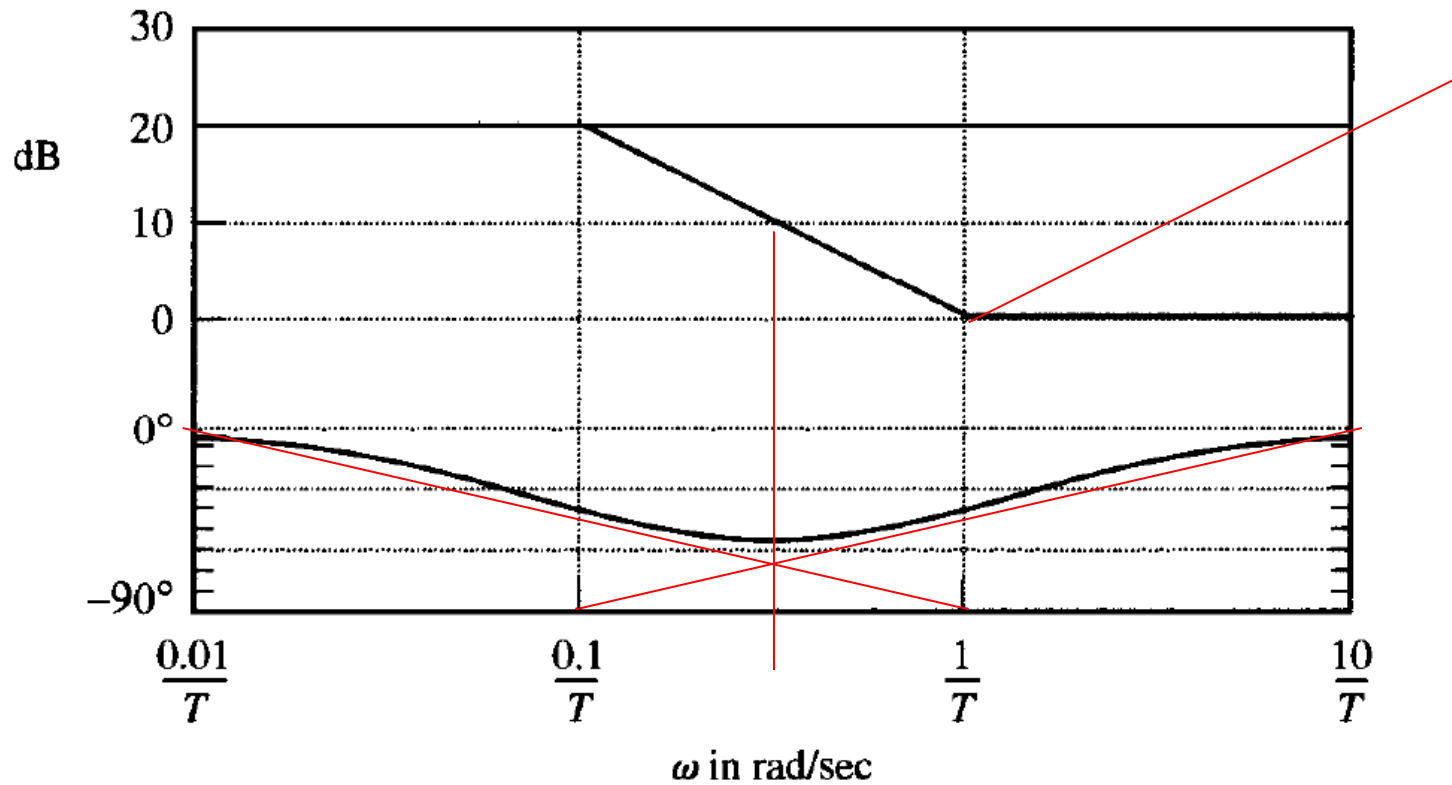


# Lag Compensator

- the polar plot is given as:



$$G_c(s) = K_c \beta \frac{Ts + 1}{\beta Ts + 1}, \quad \text{assume } K_c = 1, \beta = 10$$



- **The open-loop transfer function of the system is**

$$G_c(s)G(s) = \frac{Ts + 1}{\beta Ts + 1} KG(s) = \frac{Ts + 1}{\beta Ts + 1} G_1(s)$$

- **Define  $K_c\beta = K$ ,  $G(s) = \frac{1}{s(s + 1)(0.5s + 1)}$**
- **It is desired to compensate the system so that the static velocity error constant  $K_v$  is  $5 \text{ sec}^{-1}$ , the phase margin is at least  $40^\circ$ , gain margin is at least  $10 \text{ dB}$ .**

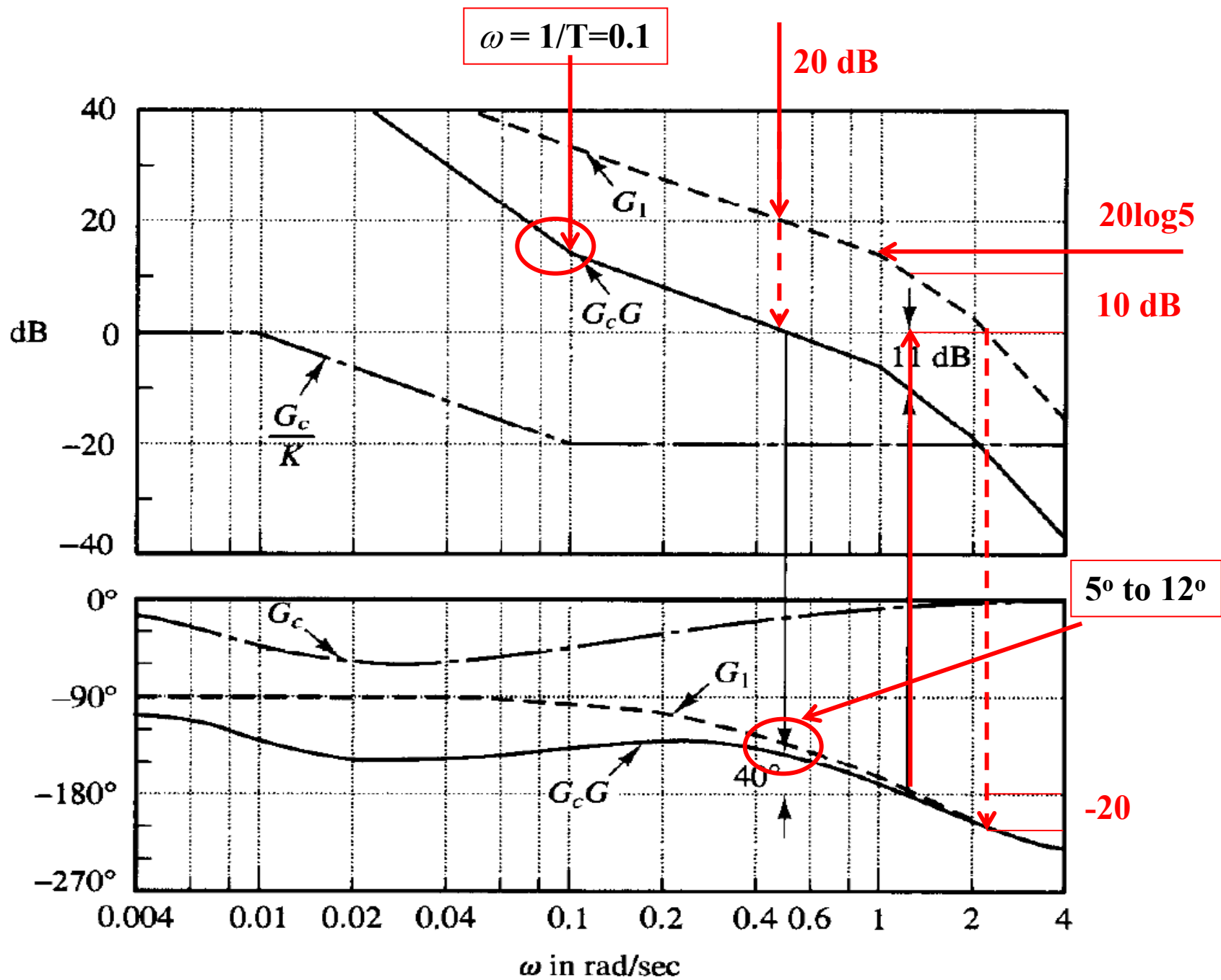
- **To meet the requirement,**

$$K_v = \lim_{s \rightarrow 0} s G_c(s) G(s) = \lim_{s \rightarrow 0} s \frac{Ts + 1}{\beta Ts + 1} G_1(s)$$

- **Therefore,** 
$$K_v = \lim_{s \rightarrow 0} \frac{sK}{s(s+1)(0.5s+1)} = K = 5$$

- **Next plot the Bode Diagram for**

$$G_1(j\omega) = KG(j\omega) = \frac{5}{j\omega(j\omega+1)(0.5j\omega+1)}$$



- The phase margin is found to be  **$-20^\circ$** , therefore unstable.
- Chose the corner frequency  $\omega = 1/T$  (which corresponds to **the zero of the lag compensator**) to be  **$0.1$  rad/sec**.
- The addition of a lag compensator will modify the phase curve of the Bode Diagram, we must allow  **$5^\circ$  to  $12^\circ$**  to compensate the this shift.
- The requirement is  **$40^\circ$** ,  **$40^\circ + 12^\circ = 52^\circ$** , therefore the phase angle is  **$-128^\circ$**  at about  **$\omega = 0.5$  rad/sec**.

- To bring the magnitude curve down to 0 dB at this new gain crossover frequency, the lag compensator must give the necessary **attenuation, -20 dB at this point** ( $\omega = 0.5$  rad/sec).
- Since  $AR = |G_1(j\omega)G_2(j\omega)\cdots| = |G_1(j\omega)||G_2(j\omega)|\cdots$
- and  $K_c\beta = K$ , therefore, **to cancel out 20 dB of  $G_1$  at ( $\omega = 0.5$  rad/sec), let**

$$20 \log \frac{1}{\beta} = -20$$

$$\therefore \beta = 10$$

- **The other corner frequency  $\omega = 1/(\beta T)$  is therefore determined as 0.01 rad/sec.**
- **Thus the transfer function of the lag compensator**

**is,**

$$G_c(s) = K_c(10) \frac{10s + 1}{100s + 1} = K_c \frac{s + \frac{1}{10}}{s + \frac{1}{100}}$$

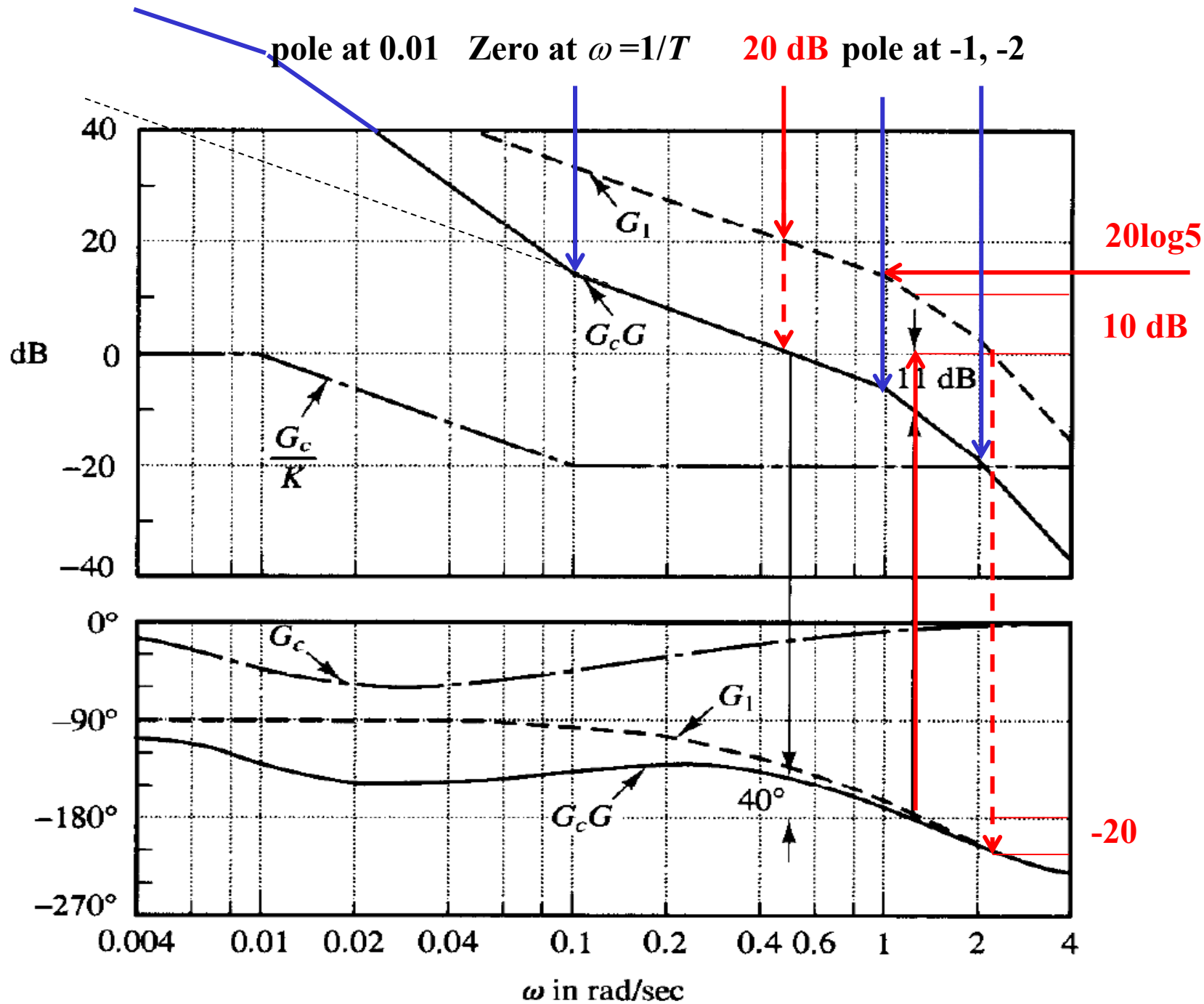
- **Since the gain  $K$  was determined to be 5 and  $\beta$  was determined to be 10,**
- **$K_c\beta = K = 5, K_c = 0.5$**



- **The open-loop transfer function of the compensated system is therefore,**

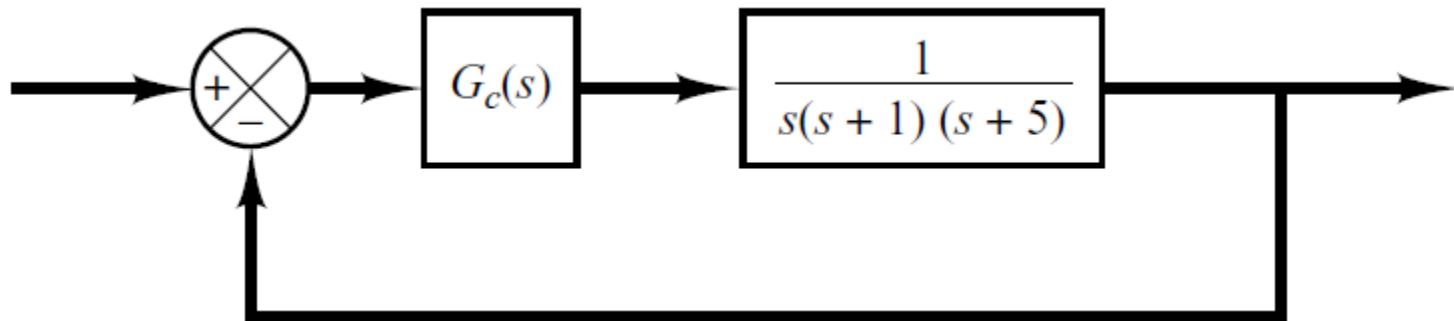
$$G_c(s)G(s) = \frac{5(10s + 1)}{s(100s + 1)(s + 1)(0.5s + 1)}$$

- **The phase margin is about 40°, gain margin is about 11dB**



# Lead-Lag

- Design a lag–lead compensator such that the static velocity error constant  $K_v$  is  $50 \text{ sec}^{-1}$  and the damping ratio  $\zeta$  of the dominant closed loop poles is 0.5. (Choose the zero of the lead portion of the lag–lead compensator to cancel the pole at  $s=-1$  of the plant.) Determine all closed-loop poles of the compensated system.



- **Choose,**

$$G_c(s) = K_c \left( \frac{s + \frac{1}{T_1}}{s + \frac{\beta}{T_1}} \right) \left( \frac{s + \frac{1}{T_2}}{s + \frac{1}{\beta T_2}} \right) = K_c \frac{(T_1 s + 1)(T_2 s + 1)}{\left( \frac{T_1}{\beta} s + 1 \right) (\beta T_2 s + 1)}$$

- **then**

$$\begin{aligned} K_v &= \lim_{s \rightarrow 0} s G_c(s) G(s) \\ &= \lim_{s \rightarrow 0} s K_c \frac{(T_1 s + 1)(T_2 s + 1)}{\left( \frac{T_1}{\beta} s + 1 \right) (\beta T_2 s + 1)} \frac{1}{s(s + 1)(s + 5)} \\ &= \frac{K_c}{5} \end{aligned}$$

- $K_V$  is  $50 \text{ sec}^{-1}$ ,  $K_C = 250$ .
- Let  $T_1 = 1$  so that  $(s+1/T_1)$  will cancel the  $(s+1)$  term of the plant, **according to the statement**
- The lead portion then becomes,  $(s+1)/(s+\beta)$ .
- The lag portion, **hope to have**

$$\left| \frac{s_1 + \frac{1}{T_2}}{s_1 + \frac{1}{\beta T_2}} \right| \cong 1, \quad -5^\circ < \angle \frac{s_1 + \frac{1}{T_2}}{s_1 + \frac{1}{\beta T_2}} < 0^\circ$$

- $s = s_1$  is one of the dominant closed-loop poles.

- Noting these requirements for the lag portion of the compensator, at  $s = s_1$ , the open-loop transfer function becomes.

$$\begin{aligned} G_c(s_1)G(s_1) &= K_c \left( \frac{s_1 + 1}{s_1 + \beta} \right) \frac{1}{s_1(s_1 + 1)(s_1 + 5)} \\ &= K_c \frac{1}{s_1(s_1 + 1)(s_1 + 5)} \end{aligned}$$

- Then at  $s = s_1$ , the following magnitude and angle conditions must be satisfied,

$$\left| K_c \frac{1}{s_1(s_1 + 1)(s_1 + 5)} \right| = 1$$

- and

$$\angle \frac{K_c}{s_1(s_1 + 1)(s_1 + 5)} = \pm 180^\circ(2k + 1)$$

- Notice  $\beta$  and  $s_1$  are still unknown.
- Given  $\zeta$  of the dominant closed-loop poles is specified as 0.5, assume (trial and error)

$$s_1 = -x + j\sqrt{3}x$$

- Substitute into the magnitude condition,

$$\left| \frac{K_c = 250}{s_1(s_1 + 1)(s_1 + 5)} \right| = 1$$

- Simplify we have,

$$x\sqrt{(\beta - x)^2 + 3x^2} \sqrt{(5 - x)^2 + 3x^2} = 125$$

$$K_C = 250$$

$$\begin{aligned} & \frac{(-x + j\sqrt{3}x)(-x + \beta + j\sqrt{3}x)(-x + 5 + j\sqrt{3}x)}{(-x + j\sqrt{3}x)(-x + \beta + j\sqrt{3}x)(-x + 5 + j\sqrt{3}x)} \\ &= -120^\circ - \tan^{-1}\left(\frac{\sqrt{3}x}{-x + \beta}\right) - \tan^{-1}\left(\frac{\sqrt{3}x}{-x + 5}\right) \\ &= -180^\circ \end{aligned}$$

$$\tan^{-1}\left(\frac{\sqrt{3}x}{-x + \beta}\right) + \tan^{-1}\left(\frac{\sqrt{3}x}{-x + 5}\right) = 60^\circ$$

- We arrive at

$$\beta = 16.025, \quad x = 1.9054$$

$$s_1 = -x + j\sqrt{3}x = -1.9054 + j3.3002$$



- Noting that the pole and zero of the lag portion of the compensator must be located near the origin, we may choose

$$\frac{1}{\beta T_2} = 0.01$$

- That is

$$\frac{1}{T_2} = 0.16025$$

- Substitute,  $T_2 = 6.25$ ,

$$\left| \frac{s_1 + \frac{1}{T_2}}{s_1 + \frac{1}{\beta T_2}} \right| = \left| \frac{-1.9054 + j3.3002 + 0.16025}{-1.9054 + j3.3002 + 0.01} \right| = 0.98 \cong 1$$

- and

$$\frac{\angle \frac{s_1 + \frac{1}{T_2}}{s_1 + \frac{1}{\beta T_2}}}{\angle \frac{-1.9054 + j3.3002 + 0.16025}{-1.9054 + j3.3002 + 0.01}} = \tan^{-1}\left(\frac{3.3002}{-1.74515}\right) - \tan^{-1}\left(\frac{3.3002}{-1.89054}\right) = -1.937^\circ$$

$$G_c(s) = 250 \left( \frac{s + 1}{s + 16.025} \right) \left( \frac{s + 0.16025}{s + 0.01} \right)$$

$$\frac{C(s)}{R(s)} = \frac{250(s + 0.16025)}{s(s + 0.01)(s + 5) + 250(s + 16.025)}$$

- closed-loop poles are at

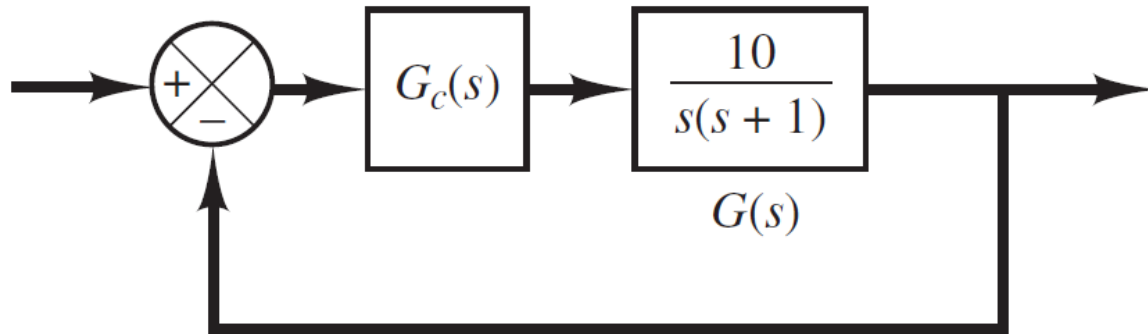
$$s = -1.8308 \pm j3.2359$$

$$s = -0.1684$$

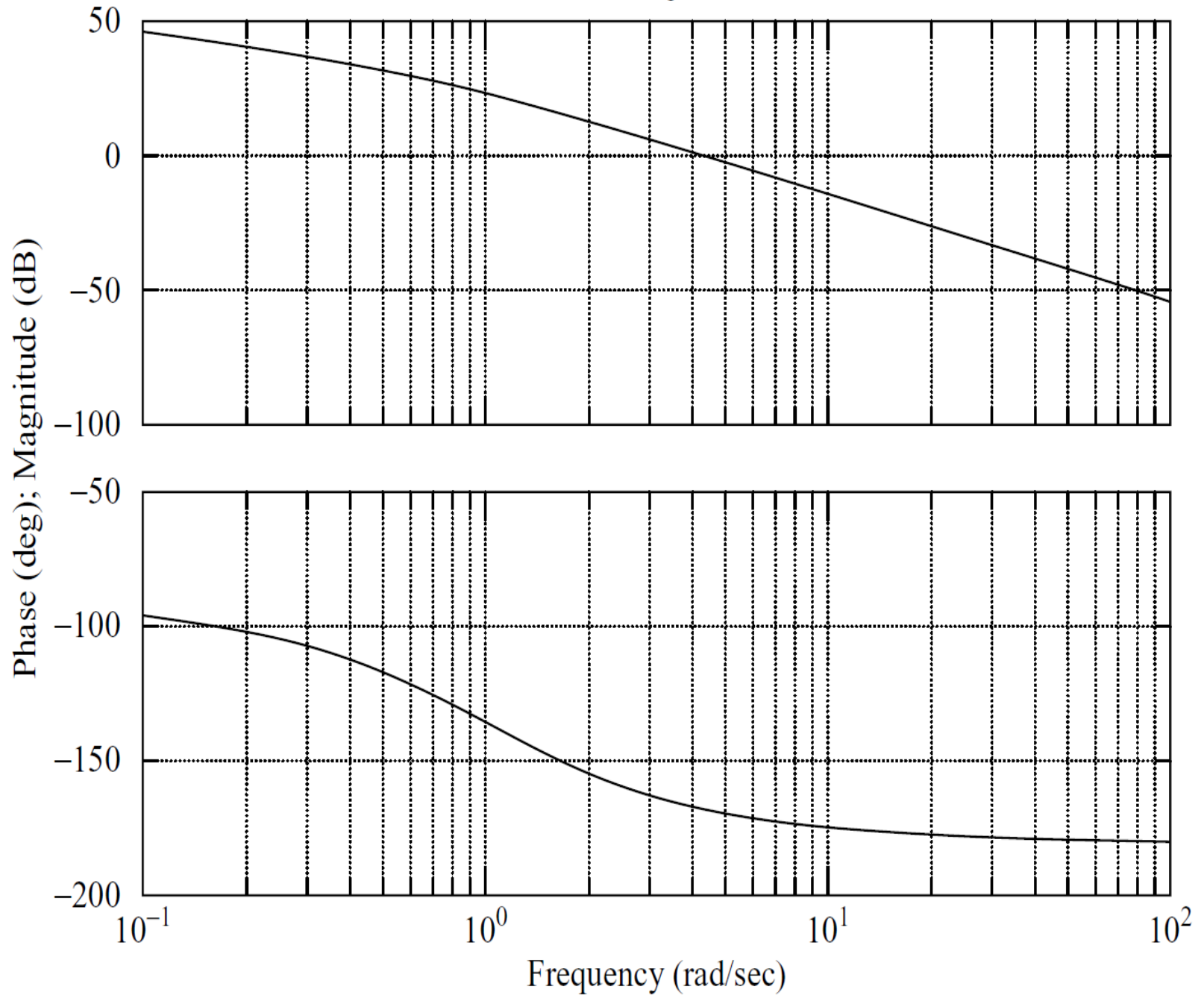
$$s = -17.205$$

# Exercise

- Consider the system shown in Figure. Design a compensator such that the closed-loop system will satisfy the requirements that the static velocity error constant  $= 20 \text{ sec}^{-1}$ , phase margin  $= 50^\circ$ , and gain margin  $G 10 \text{ dB}$ .



Bode Diagram of  $G_1(s) = 20/[s(s + 1)]$



- Since the specification calls for a phase margin of  $50^\circ$ , the additional phase lead necessary to
- satisfy the phase-margin requirement is  $36^\circ$ . A lead compensator can contribute this amount.
- Taking the shift of the gain crossover frequency into consideration, we may assume that  $\phi_m$ , the maximum phase lead required, is approximately  $41^\circ$ .
- $\phi_m=41^\circ$  corresponds to  $\alpha=0.2077$ . Note that  $\alpha=0.21$  corresponds to  $\phi_m=40.76^\circ$ .
- let us choose  $\alpha=0.21$ .

- The amount of the modification in the magnitude curve at  $\omega = 1/\sqrt{\alpha}T$  due to the inclusion of the compensator,  $(Ts+1)/(\alpha Ts+1)$ , is

$$\left| \frac{1 + j\omega T}{1 + j\omega \alpha T} \right|_{\omega = \frac{1}{\sqrt{\alpha}T}} = \left| \frac{1 + j \frac{1}{\sqrt{\alpha}}}{1 + j\alpha \frac{1}{\sqrt{\alpha}}} \right| = \frac{1}{\sqrt{\alpha}}$$

$$\frac{1}{\sqrt{\alpha}} = \frac{1}{\sqrt{0.21}} = 6.7778 \text{ dB}$$

- From this diagram, we find the frequency point where  $G_1(j\omega) = -6.7778$  dB occurs at  $\omega = 6.5686$  rad/sec,

$$\omega_c = \frac{1}{\sqrt{\alpha}T}$$

$$\frac{1}{T} = \omega_c \sqrt{\alpha} = 6.5686 \sqrt{0.21} = 3.0101$$

$$\frac{1}{\alpha T} = \frac{\omega_c}{\sqrt{\alpha}} = \frac{6.5686}{\sqrt{0.21}} = 14.3339$$

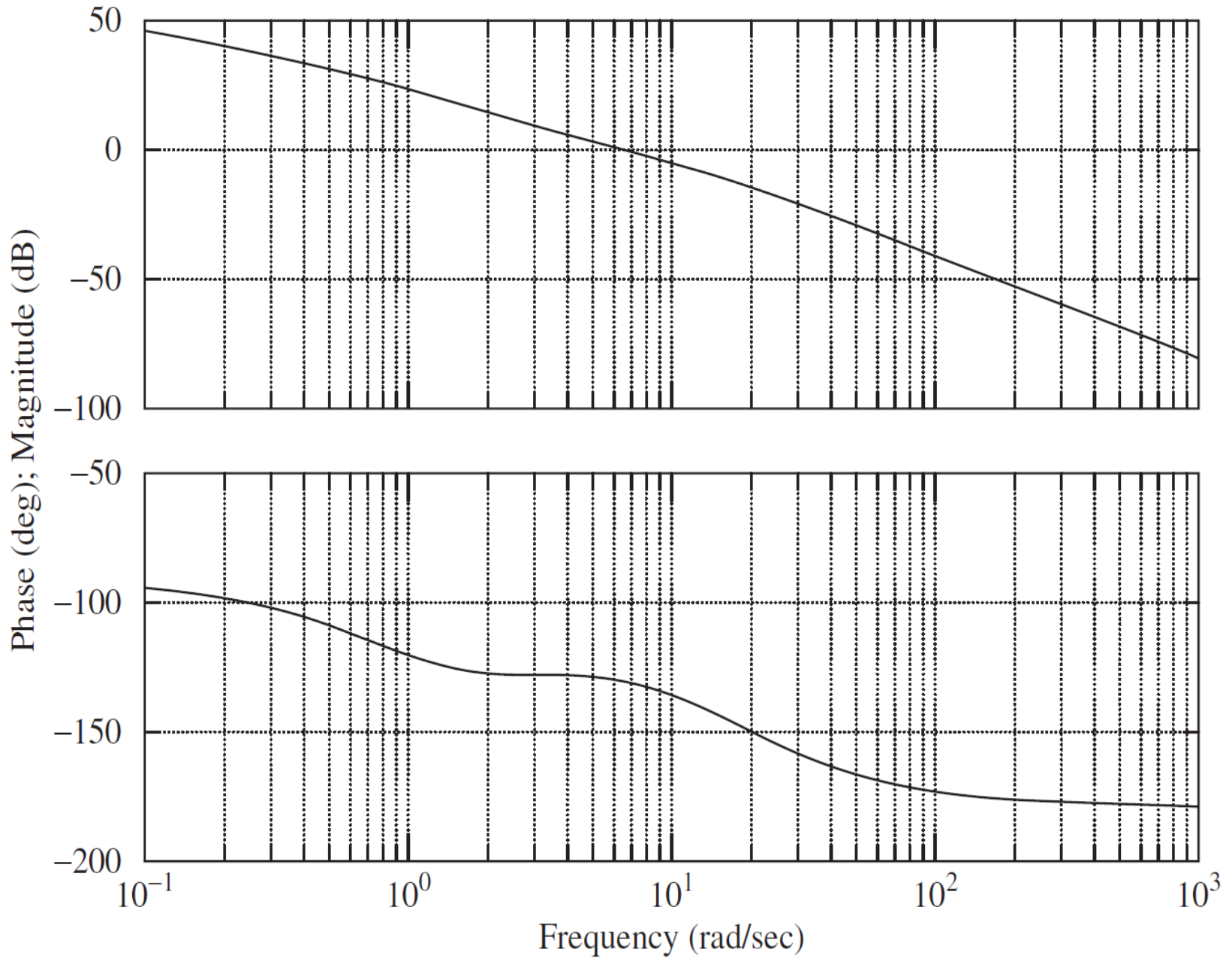


$$G_c = K_c \frac{s + 3.0101}{s + 14.3339} = K_c \alpha \frac{0.3322s + 1}{0.06976s + 1}$$

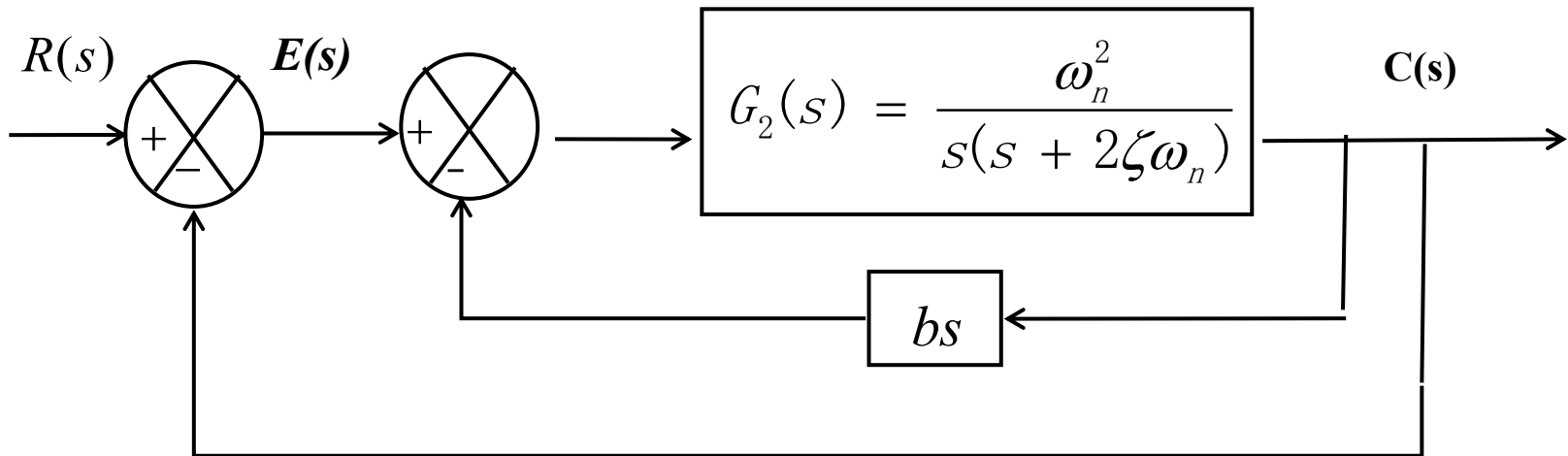
$$K_c = \frac{K}{\alpha} = \frac{2}{0.21} = 9.5238$$

$$G_c = 9.5238 \frac{s + 3.0101}{s + 14.3339} = 2 \frac{0.3322s + 1}{0.06976s + 1}$$

Bode Diagram of  $G_c(s)G(s)$



# Extra Homework



$$G(s) = \frac{16}{s(s + 4)}$$

- $b = 0$ , determine the damping ratio undamped natural frequency, peak overshoot from a unit step input and the steady-state error resulting from a unit ramp input.
- Determine  $b$  which will increase the equivalent damping ratio of the system to 0.8
- The resulting steady-state error from unit ramp input.