Smith's Method

Assumed model:

$$G(s) = \frac{Ke^{-\theta s}}{\tau^2 s^2 + 2\zeta \tau s + 1}$$

If the original process transfer function contains a time delay

$$t = t' - \theta$$

Performance Criteria

Performance of a Feedback Control System

- In the early days
 - Stability
 - Static Accuracy
- Modern Complex Control System Also Interested in
 - Sensitivity
 - Transient Response
 - Residual Noise Jitter

Stability s-plane 0 < ζ < **1** c(t) ζ = 0 c(t)X - 1 Underdamped **σ < 0** $\sigma > 0$ $\zeta \ge 1$ c(t)Undamped + 1 Overdamped σ



Sensitivity

Definition

$$S_{K}^{H}(s) = \frac{d \ln H(s)}{d \ln K(s)} = \frac{\% \text{ change in } H(s)}{\% \text{ change in } K(s)}$$

where
$$H(s) = \frac{C(s)}{R(s)}$$

Rearrange

$$S_K^H(s) = \frac{dH(s)/H(s)}{dK(s)/K(s)}$$



$$H(s) = \frac{K_1(s)G(s)}{1 + K_2(s)G(s)}$$

• Sensitivity of *H*(s) with respect to *K*₁

$$S_{K_1}^H(s) = \frac{dH(s)/H(s)}{dK_1(s)/K_1(s)} = \frac{K_1(s)}{H(s)}\frac{dH(s)}{dK_1(s)}$$

where

$$\frac{dH(s)}{dK_{1}(s)} = \frac{G(s)}{1 + K_{2}(s)G(s)} = \frac{H(s)}{K_{1}(s)}$$

Therefore,

$$S_{K_1}^H(s) = \frac{dH(s) / H(s)}{dK_1(s) / K_1(s)} = 1$$

• Sensitivity of H(s) with respect to K_2

$$S_{K_2}^{H}(s) = \frac{dH(s)/H(s)}{dK_2(s)/K_2(s)} = \frac{K_2(s)}{H(s)}\frac{dH(s)}{dK_2(s)}$$

- where

$$\frac{dH(s)}{dK_2(s)} = \frac{0 - K_1 G^2(s)}{\left[1 + K_2(s)G(s)\right]^2}$$
$$= \frac{-K_2(s)G^2(s)}{K_1(s)\left[1 + K_2(s)G(s)\right]^2}$$

- Therefore

$$S_{K_{2}}^{H}(s) = \frac{K_{2}(s)}{H(s)} \frac{-K_{1}^{2}G^{2}(s)}{K_{1}(s)[1+K_{2}(s)G(s)]^{2}}$$
$$= \frac{-K_{2}(s)}{H(s)} \frac{H^{2}(s)}{K_{1}(s)} = \frac{-K_{2}(s)G(s)}{1+K_{2}(s)G(s)}$$

– When $K_2G(s) >> 1$

$$S_{K_2}^{H}(s) = \frac{-K_2(s)G(s)}{1+K_2(s)G(s)} \approx -1$$

• Sensitivity of *H*(s) with respect to *G*(s)

$$S_G^H(s) = \frac{dH(s)/H(s)}{dG(s)/G(s)} = \frac{G(s)}{H(s)}\frac{dH(s)}{dG(s)}$$

where

$$\frac{dH(s)}{dG(s)} = \frac{\left[1 + K_2(s)G(s)\right]K_1 - K_1(s)G(s)K_2(s)}{\left[1 + K_2(s)G(s)\right]^2}$$
$$= \frac{K_1(s)}{\left[1 + K_2(s)G(s)\right]^2}$$

Therefore

$$S_{G(s)}^{H}(s) = \frac{G(s)}{H(s)} \frac{K_{1}(s)}{\left[1 + K_{2}(s)G(s)\right]^{2}}$$
$$= \frac{1}{1 + K_{2}(s)G(s)}$$

Desire to have

 $1 + K_2(s)G(s) >> 1$



• The transfer function,

$$\frac{E(s)}{R(s)} = \frac{1}{1 + G(s)}$$

• Apply the final-value theorem of the Laplace transform,

$$e_{ss} = \lim_{t \to \infty} e(t) = \lim_{s \to 0} \frac{sR(s)}{1 + G(s)}$$

- We usually interested in,
 - Position input
 - Velocity input
 - Acceleration input

These inputs are usually called
 – Unit step

$$r(t) = U(t) \qquad R(s) = 1/s$$

– Unit ramp

$$r(t) = tU(t) \qquad R(s) = 1/s^2$$

– Paraboloid

$$r(t) = \frac{1}{2}t^2U(t)$$
 $R(s) = 1/s^3$

• Assume the loop transfer function G(s),

$$G(s) = \frac{K(1+T_1s)(1+T_2s)\cdots(1+T_ms)}{s^n [(T_0s)^2 + 2\zeta\omega_n s + 1](1+T_as)(1+T_bs)\cdots(1+T_q)}$$

 Based on the pure integrations in the denominator of the open loop transfer function, the system is defined as the nth type system

- As the system type is increased, the accuracy is improved. However,
- in practice, we usually never design a control system greater than type 2 because it is very difficult to stabilize a control system containing more than two pure integrations.
- There will be a tradeoff between them

- Unit step $e_{ss} = \lim_{s \to 0} \frac{s(1/s)}{1 + G(s)} = \frac{1}{1 + \lim_{s \to 0} G(s)}$
- Define position constant

• therefore
$$K_{p} = \lim_{s \to 0} G(s)$$
$$e_{ss} = \frac{1}{1 + K_{p}}$$



Define velocity constant

$$\mathbf{e} \qquad K_{v} = \lim_{s \to 0} sG(s)$$
$$\mathbf{e} \qquad e_{ss} = \frac{1}{K_{v}}$$

therefore

- Parabolic $e_{ss} = \lim_{s \to 0} \frac{s(1/s^3)}{1 + G(s)} = \frac{1}{\lim_{s \to 0} s^2 G(s)}$
- Define acceleration constant

therefore
$$K_a = \lim_{s \to 0} s^2 G(s)$$

 $e_{ss} = \frac{1}{K_a}$

• If the input composed of all three forms,

$$r(t) = AU(t) + BtU(t) + \frac{1}{2}Ct^{2}U(t)$$

– No pure integration

$$e_{ss} = \frac{1}{1 + K_p} + \infty + \infty$$

– One pure integration

$$e_{ss} = 0 + \frac{B}{K_v} + \infty$$

- Two pure integration

$$e_{ss} = 0 + 0 + \frac{C}{K_a}$$

- Relationship between static error constants to closed-loop poles and zeros
 - A close loop transfer function,

 $\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)}$

- The relation between the input and error,

$$\frac{E(s)}{R(s)} = \frac{1}{1 + G(s)}$$

$$E(s) = R(s) - C(s)$$

- It is evident that

$$\frac{C(s)}{R(s)} = 1 - \frac{E(s)}{R(s)}$$

– Assume C(s)/R(s) is as

$$\frac{C(s)}{R(s)} = \frac{K(s+z_1)(s+z_2)\cdots(s+z_m)}{(s+p_1)(s+p_2)\cdots(s+p_n)} = \frac{K\Pi_{j=1}^m(s+z_j)}{\Pi_{j=1}^n(s+p_j)}$$

– And expend E(s)/R(s) as power series in s

$$\frac{E(s)}{R(s)} = \frac{1}{1+G(s)} = \frac{1}{1+K_p} + \frac{1}{K_v}s + \frac{1}{K_a}s^2 + \cdots$$

- Position Constant
 - Let s approach 0,

$$\lim_{s \to 0} \frac{E(s)}{R(s)} = \lim_{s \to 0} \frac{1}{1 + G(s)} = \frac{1}{1 + K_p}$$

-therefore $\frac{C(0)}{R(0)} = 1 - \frac{E(0)}{R(0)} = \frac{K_p}{1 + K_p}$

– Solving for K_p in terms of C(s)/R(s)

$$K_p = \frac{C(0) / R(0)}{1 - C(0) / R(0)}$$

$$\frac{C(0)}{R(0)} = \frac{K(0+z_1)(0+z_2)\cdots(0+z_m)}{(0+p_1)(0+p_2)\cdots(0+p_n)} = \frac{K\Pi_{j=1}^m z_j}{\Pi_{j=1}^n p_j}$$

- therefore

$$K_{p} = \frac{K \prod_{j=1}^{m} z_{j}}{\prod_{j=1}^{n} p_{j} - K \prod_{j=1}^{m} z_{j}}$$

Velocity Constant

$$\frac{C(s)}{R(s)} = 1 - \frac{E(s)}{R(s)}$$



- Take derivative with respect to s

$$\frac{d}{ds} \left(\frac{C(s)}{R(s)} \right) = -\frac{1}{K_v}$$

- Therefore,



– and

$$\frac{1}{K_{v}} = -\left\{\frac{d}{ds}\left[\ln K + \ln(s+z_{1}) + \dots + \ln(s+z_{m}) - \ln K - \ln(s+p_{1}) - \dots - \ln(s+p_{n})\right]\right\}_{s=0}$$

It may be simplified as



- or



Acceleration Constant

$$\frac{d^2}{ds^2} \left[\ln \frac{C(s)}{R(s)} \right] = \frac{\frac{d^2}{ds^2} \left[\frac{C(s)}{R(s)} \right]}{\frac{C(s)}{R(s)}} - \left\{ \frac{\frac{d}{ds} \left[\frac{C(s)}{R(s)} \right]}{\frac{C(s)}{R(s)}} \right\}^2$$

– recall

$$\frac{C(s)}{R(s)} = 1 - \frac{E(s)}{R(s)} = 1 - \left(\frac{1}{1+K_p} + \frac{1}{K_v}s + \frac{1}{K_a}s^2 + \cdots\right)$$

- Therefore

$$\left\{\frac{d^2}{ds^2}\left[\ln\frac{C(s)}{R(s)}\right]\right\}_{s=0} = -\frac{1}{K_v^2} - \frac{2}{K_a}$$

– Recall

$$-\left[\frac{d}{ds}\ln\frac{C(s)}{R(s)}\right]_{s=0} = \frac{1}{K_v} = \sum_{j=1}^n \frac{1}{p_j} - \sum_{j=1}^m \frac{1}{z_j}$$

– Therefore

$$\sum_{j=1}^{n} \frac{1}{p_{j}^{2}} - \sum_{j=1}^{m} \frac{1}{z_{j}^{2}} = -\frac{1}{K_{v}^{2}} - \frac{2}{K_{a}}$$

• Example

$$\frac{C(s)}{R(s)} = \frac{K_m / T_m}{s^2 + (1 / T_m)s + K_m / T_m}$$

Where K_m is the system gain and T_m is the time constant of the open-loop transfer function



Or in the more familiar form

$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

• From the definition,

$$K_{v} = \lim_{s \to 0} sG(s) = \lim_{s \to 0} s \frac{K_{m}}{s(T_{m}s+1)} = K_{m}$$

• therefore,

$$K_{v} = K_{m} = \frac{\omega_{n}}{2\zeta}$$

This is of great importance, to have very accurate response to velocity response, ζ must be very small—in inertial navigation applications, such as missile control.

• General form,

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$
$$= \frac{A_1 s^n + A_2 s^{n-1} + \dots + A_n s + A_{n+1}}{B_1 s^m + B_2 s^{m-1} + \dots + B_m s + B_{m+1}}$$

Recall

$$\frac{E(s)}{R(s)} = \frac{1}{1+G(s)} = \frac{1}{1+K_p} + \frac{1}{K_v}s + \frac{1}{K_a}s^2 + \cdots$$

Therefore

$$e_{ss} = \frac{r(t)}{1+K_p} + \frac{\dot{r}(t)}{K_v} + \frac{\ddot{r}(t)}{K_a} + \cdots$$

 It has been proved that for e_{ss} = 0, with a step input,

$$\frac{1}{1+K_p}$$
 is a function of $\mathbf{B}_{m+1} - \mathbf{A}_{n+1}$, and $\mathbf{B}_{m+1} = \mathbf{A}_{n+1}$

• If, $\frac{C(s)}{R(s)} = \frac{B_{m+1}}{B_1 s^m + B_2 s^{m-1} + \dots + B_m s + B_{m+1}}$

The zero steady-state step error system

• Example (one pure integrator)





A zero steady-state step error system (prove it)

 It has also been proved that for e_{ss} = 0, with a ramp input,

 $\frac{1}{K_{v}}$ is a function of $B_{m+1} - A_{n+1}, B_{m} - A_{n},$ and

$$B_{m+1} = A_{n+1} \text{ and } B_m = A_n,$$

It can be shown that for ramp input,

$$\frac{C(s)}{R(s)} = \frac{B_m s + B_{m+1}}{B_1 s^m + B_2 s^{m-1} + \dots + B_m s + B_{m+1}}$$

is a zero steady-state ramp error system



It contains two pure integrates

$$\frac{C(s)}{R(s)} = \frac{\frac{T_1 s + 1}{T_2 s + 1} \frac{K}{s^2}}{1 + \frac{T_1 s + 1}{T_2 s + 1} \frac{K}{s^2}} = \frac{K(T_1 s + 1)}{T_2 s^2 + s^2 + KT_1 s + K}$$

 It is a zero steady-state ramp error system