# **Bode Diagram**

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# Frequency Response

- Frequency Response is the steadystate behavior of the system when forced by a sinusoidal input.
- Consider a first order system

$$G(s) = \frac{C(s)}{R(s)} = \frac{K_p}{\tau s + 1}$$

• Let us assume that this process is subjected to a sinusoidal input

$$u(t) = A \sin \omega t$$

where A is the amplitude and  $\omega$  is the frequency

Laplace transform of the input gives

$$U(s) = \frac{A\omega}{s^2 + \omega^2}$$

• hence, the Laplace transform of the output becomes  $K = A \omega$ 

$$C(s) = \frac{K_p}{\tau s + 1} \frac{A\omega}{s^2 + \omega^2}$$

expanded into fractions

$$C(s) = \frac{B_1}{s + 1/\tau} + \frac{B_2}{s + j\omega} + \frac{B_3}{s - j\omega}$$

• inverse Laplace transform, take  $t \to \infty$ , steadystate

$$u(t \to \infty) = \frac{AK_p}{\omega^2 \tau^2 + 1} \sin \omega t - \frac{AK_p}{\omega^2 \tau^2 + 1} \cos \omega t$$

- Or  $a_1 \sin \omega t + a_2 \cos \omega t = a_3 \sin(\omega t + \varphi)$
- where  $a_3 = \sqrt{a_1^2 + a_2^2}, \varphi = \tan^{-1} \left(\frac{a_2}{a_1}\right)$
- Therefore,  $u(t \to \infty) = \frac{AK_p}{\sqrt{\omega^2 \tau^2 + 1}} \sin(\omega t + \varphi)$  $\varphi = \tan^{-1}(-\omega \tau)$

- It can be seen that the response to a sinusoidal input signal is a sinusoidal signal with the same frequency but a different angle. The output signal lags behind the input signal by an angle  $\varphi$ , which depends on the frequency  $\omega$ .
- The ratio between the amplitude of the input sine wave and the output sine wave, called amplitude ratio, K

$$AR = \frac{K_p}{\sqrt{\omega^2 \tau^2 + 1}}$$

• Substitution of  $s = j\omega$  in the transfer function G(s) results in a transfer function in the frequency domain  $G(j\omega)$ . Then the magnitude is equal to the amplitude ratio AR:

$$AR = |G(j\omega)|$$

and the phase angle or argument is equal to:

$$\varphi = \arg[G(j\omega)]$$

•  $G(j\omega)$  is a complex number, it can therefore be represented by a real and imaginary part:

$$G(j\omega) = \text{Re}[G(j\omega)] + j \text{Im}[G(j\omega)]$$

for which:

$$AR = |G(j\omega)| = \sqrt{(\text{Re}[G(j\omega)])^2 + (j \text{Im}[G(j\omega)])^2}$$

$$\varphi = \arg[G(j\omega)] = \tan^{-1} \frac{\operatorname{Im}[G(j\omega)]}{\operatorname{Re}[G(j\omega)]}$$

• When substituting  $s = j\omega$  into

$$G(j\omega) = \frac{C(j\omega)}{R(j\omega)} = \frac{K_p}{\tau(j\omega) + 1} \frac{(-\tau(j\omega) + 1)}{(-\tau(j\omega) + 1)}$$

$$=\frac{K_p}{\omega^2 \tau^2 + 1} - j \frac{K_p \omega \tau}{\omega^2 \tau^2 + 1}$$

• Therefore,  $magnitude = AR = \frac{K_p}{\sqrt{\omega^2 \tau^2 + 1}}$ 

phase angle 
$$\varphi = \tan^{-1}(-\omega\tau)$$

 A transfer function is often a combination of subtransfer functions, consisting of numerator and denominator terms:

$$G(j\omega) = \frac{G_1(j\omega)G_2(j\omega)\cdots G_n(j\omega)}{G_{n+1}(j\omega)G_{n+2}(j\omega)\cdots G_m(j\omega)}$$

It can easily be shown that

$$AR = |G_1(j\omega)G_2(j\omega)\cdots| = |G_1(j\omega)|G_2(j\omega)\cdots$$

$$\varphi = \arg[G_1(j\omega)G_2(j\omega)\cdots] = \arg[G_1(j\omega)] + \arg[G_2(j\omega)] + \cdots$$

 For inverse transfer functions the amplitude ratio and phase angle can be derived from the property:

$$AR = \left| \frac{1}{G(j\omega)} \right| = \frac{1}{|G(j\omega)|}$$
  $\varphi = \arg \frac{1}{G(j\omega)} = -\arg [G(j\omega)]$ 

Therefore,
$$AR = |G(j\omega)| = \frac{\prod_{i=1}^{n} |G_i(j\omega)|}{\prod_{j=n+1}^{m} |G_j(j\omega)|}$$

$$\varphi = \arg[G(j\omega)] = \sum_{i}^{n} \arg[G_{i}(j\omega)] - \sum_{j=n+1}^{m} \arg[G_{j}(j\omega)]$$

# **Bode Diagram**

- The graphs in which the amplitude ratio and phase shift are plotted as a function of the frequency ω, are called Bode diagrams.
- In the Bode plot,  $\log(AR)$  and  $\varphi$  are shown as a function of  $\omega$ . In case of the first-order process,

$$magnitude = AR = \frac{K_p}{\sqrt{\omega^2 \tau^2 + 1}}$$

• Or 
$$\log(AR) = \log(K_p) - \frac{1}{2}\log(\omega^2\tau^2 + 1)$$

- thus  $\log (AR)$  becomes a linear function of  $\log(\omega \tau)$  with a slope of -1.
- In case of  $\omega << 1/\tau$ :

$$\log(AR) = \log(K_p) + \frac{1}{2}\log(1) = \log(K_p)$$
 Slope = 0

thus the gain of the process is independent of the frequency  $\omega$ .

• In case  $\omega = 1/\tau$ :

$$\log(AR) = \log(K_p) + \frac{1}{2}\log(2)$$

• In case of  $\omega >> 1/\tau$ :

Slope = 
$$-1$$

$$\log(AR) = \log(K_p) - \frac{1}{2}\log(\omega^2\tau^2 + 1) \approx \log(K_p) - \log\omega - \log\tau$$

*phase angle* 
$$\varphi = \tan^{-1}(-\omega\tau)$$

• In case of  $\omega >> 1/\tau$ :

phase angle 
$$\varphi = \tan^{-1}(\infty) = -90^{\circ}$$

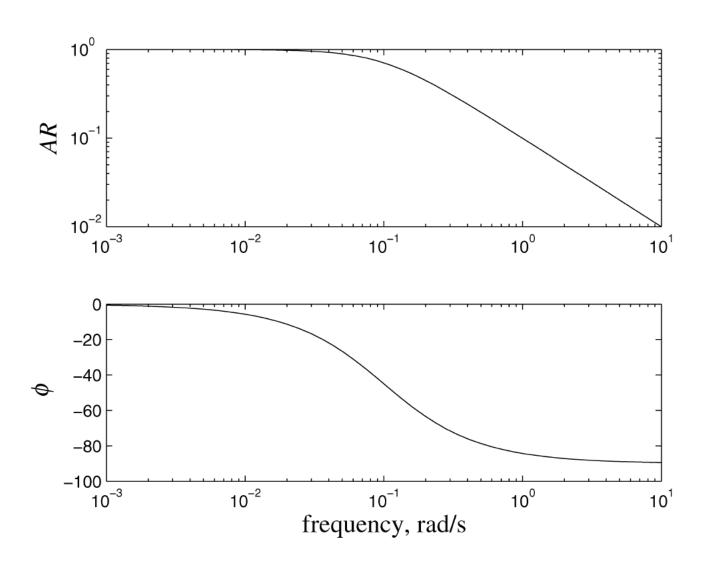
• In case  $\omega \ll 1/\tau$ :

phase angle 
$$\varphi = \tan^{-1}(0) = 0^{\circ}$$

• In case  $\omega = 1/\tau$ :

phase angle 
$$\varphi = \tan^{-1}(1) = -45^{\circ}$$

### Bode diagram of a first-order process



### Second-order Non-interacting System

$$G(s) = \frac{K_1}{(\tau_1 s + 1)} \frac{K_2}{(\tau_2 s + 1)}$$

the amplitude ratio and phase angle become:

$$AR = |G(j\omega)| = \frac{\prod_{i=1}^{n} |G_i(j\omega)|}{\prod_{j=n+1}^{m} |G_j(j\omega)|} \qquad AR = \frac{K_1 K_2}{\sqrt{\omega^2 \tau_1^2 + 1} \sqrt{\omega^2 \tau_2^2 + 1}}$$

$$\varphi = \arg[G(j\omega)] = \sum_{i=1}^{n} \arg[G_i(j\omega)] - \sum_{j=n+1}^{m} \arg[G_j(j\omega)]$$

$$\varphi = \tan^{-1}(-\omega\tau_1) + \tan^{-1}(-\omega\tau_2)$$

## Underdamped Second-order System

Second-order System

$$G(s) = \frac{K_p}{\tau^2 s^2 + 2\zeta \tau s + 1}$$

• Substituting  $s = j\omega$  into equation and rearranging results in:

$$G(j\omega) = \frac{K_p}{(1-\omega^2\tau^2) + 2j\zeta\omega\tau}$$

Recall:

$$AR = |G(j\omega)| = \sqrt{(\text{Re}[G(j\omega)])^2 + (j \text{Im}[G(j\omega)])^2}$$

$$\varphi = \arg[G(j\omega)] = \tan^{-1} \frac{\text{Im}[G(j\omega)]}{\text{Re}[G(j\omega)]}$$

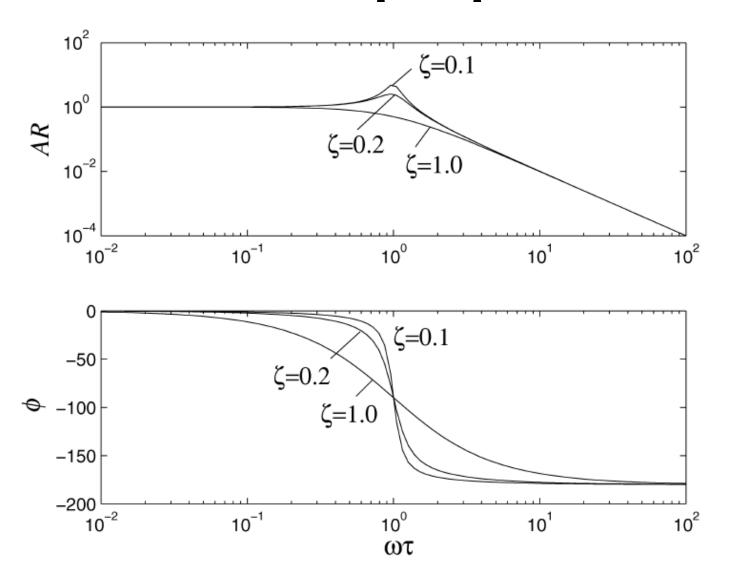
• Therefore, 
$$G(j\omega) = \frac{K_p}{(1-\omega^2\tau^2) + 2j\zeta\omega\tau} \frac{(1-\omega^2\tau^2) - 2j\zeta\omega\tau}{(1-\omega^2\tau^2) - 2j\zeta\omega\tau}$$

$$=\frac{K_p}{\left(1-\omega^2\tau^2\right)^2+\left(2\zeta\omega\tau\right)^2}\left[\left(1-\omega^2\tau^2\right)-2j\zeta\omega\tau\right]$$

$$AR = \frac{K_p}{\sqrt{(1 - \omega^2 \tau^2)^2 + (2\zeta\omega\tau)^2}}$$

$$\varphi = \tan^{-1} \left( -\frac{2\zeta\omega\tau}{1 - \omega^2\tau^2} \right)$$

# Bode diagram for second-order under-damped process



 On the Bode magnitude plot, decibels are used, defined as:

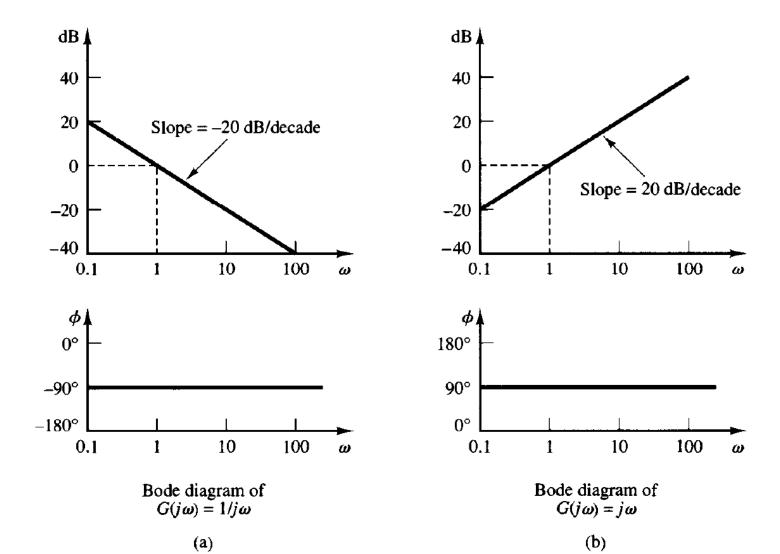
$$a_{dB} = 20 \log_{10} a$$
, decibels or dB

• The log-magnitude curve for a constant gain K is a horizontal straight line at the magnitude of

$$20\log_{10} K$$
,  $dB$ 

• Integral and derivative factors  $(j\omega)^{\mp}$ 

$$20\log|(j\omega)^{\mp}| = \mp 20\log\omega, \ dB$$



# **Example I**

For the transfer function:

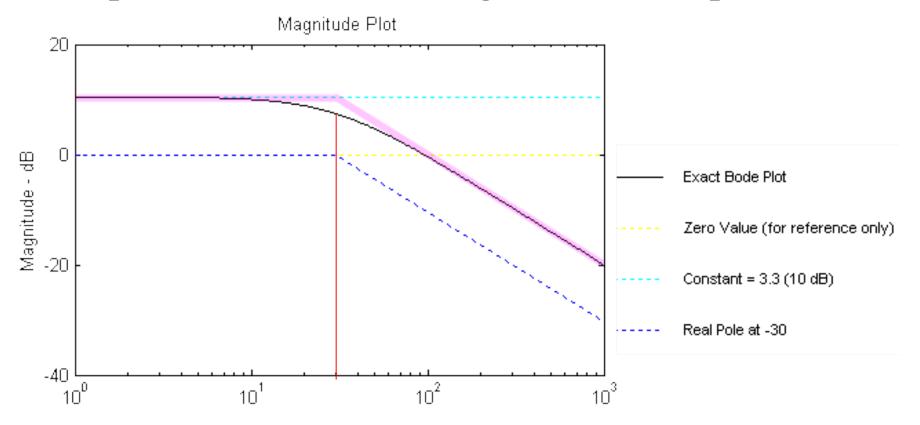
$$H(s) = \frac{100}{s + 30}$$

• Step 1: Rewrite the transfer function in proper form. Make both the lowest order term in the numerator and denominator unity. Therefore, the numerator is an order 0 polynomial, the denominator is order 1.

$$H(s) = \frac{100}{30} \frac{1}{\frac{s}{30} + 1} = 3.3 \frac{1}{\frac{s}{30} + 1}$$

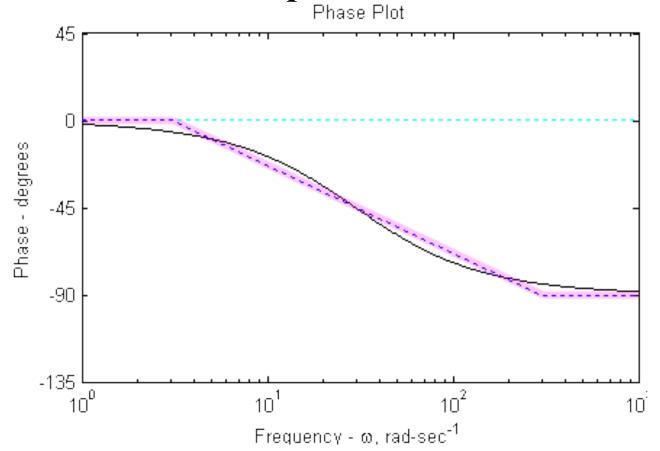
- Step 2: Separate the transfer function into its constituent parts.
- The transfer function has 2 components:
  - A constant of 3.3
  - A pole at s=-30

#### • Step 3: Draw the Bode diagram for each part.



- The constant is the cyan (青色) line (A quantity of 3.3 is equal to 10.4 dB). The phase is constant at 0 degrees.
- The pole at -30 rad/sec is the blue line. It is 0 dB up to the break frequency, then drops off with a slope of -20 dB/dec.
- The phase is 0 degrees up to 1/10 the break frequency (3 rad/sec) then drops linearly down to -90 degrees at 10 times the break frequency (300 rad/sec).

Step 4: Draw the overall Bode diagram by adding up the results from step 3.



# **Example II**

Bode Diagram for the transfer function:

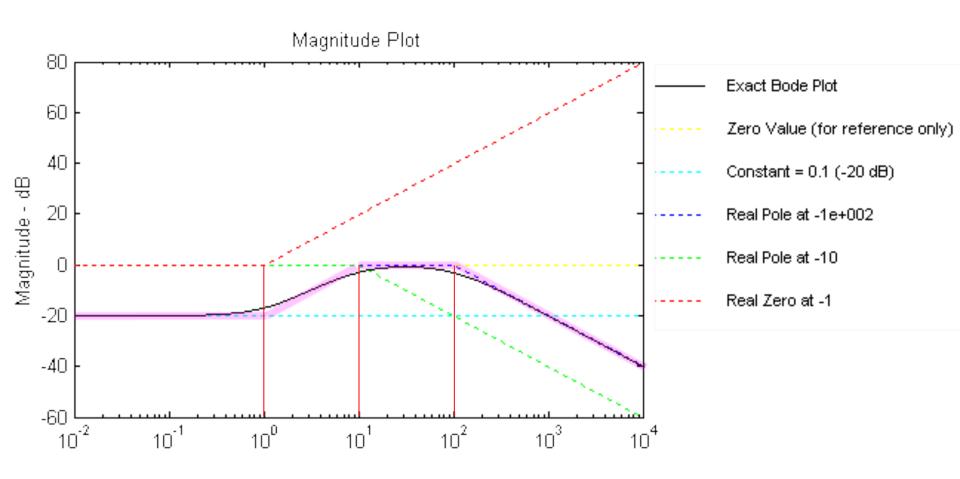
$$H(s) = 100 \frac{s+1}{(s+10)(s+100)} = 100 \frac{s+1}{s^2+110s+1000}$$

• Step 1: Rewrite the transfer function in proper form. Make both the lowest order term in the numerator and denominator unity. The numerator is an order 1 polynomial, the denominator is order 2.

$$H(s) = \frac{100}{10 \cdot 100} \frac{\frac{s}{1} + 1}{\left(\frac{s}{10} + 1\right)\left(\frac{s}{100} + 1\right)} = 0.1 \frac{\frac{s}{1} + 1}{\left(\frac{s}{10} + 1\right)\left(\frac{s}{100} + 1\right)}$$

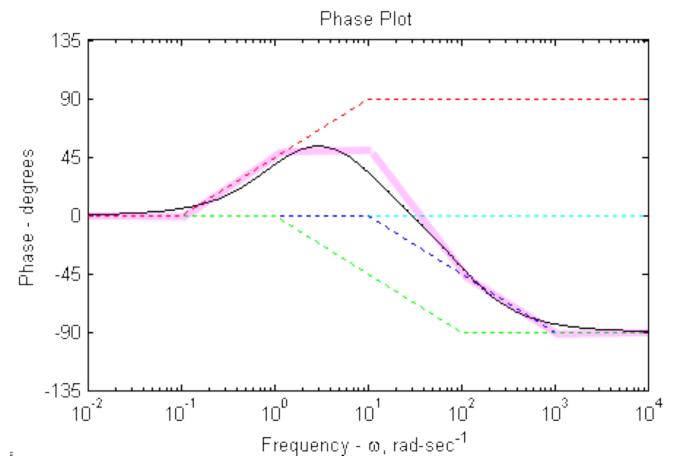
- Step 2: Separate the transfer function into its constituent parts.
- The transfer function has 4 components:
  - A constant of 0.1
  - A pole at s=-10
  - **− A pole at s=-100**
  - A zero at s=-1

- Step 3: Draw the Bode diagram for each part.
- The constant is the cyan line (A quantity of 0.1 is equal to -20 dB). The phase is constant at 0 degrees.
- The zero at 1 rad/sec is the red line. It is 0 dB up to the break frequency, then rises at 20 dB/dec. The phase is 0 degrees up to 1/10 the break frequency (0.1 rad/sec) then rises linearly to 90 degrees at 10 times the break frequency (10 rad/sec).



- The pole at 10 rad/sec is the green line. It is 0 dB up to the break frequency, then drops off with a slope of -20 dB/dec. The phase is 0 degrees up to 1/10 the break frequency (1 rad/sec) then drops linearly down to -90 degrees at 10 times the break frequency (100 rad/sec).
- The pole at 100 rad/sec is the blue line. It is 0 dB up to the break frequency, then drops off with a slope of -20 dB/dec. The phase is 0 degrees up to 1/10 the break frequency (10 rad/sec) then drops linearly down to -90 degrees at 10 times the break frequency (1000 rad/sec).

Step 4: Draw the overall Bode diagram by adding up the results from step 3.



# **Bode Diagram Example**

Bode diagram of

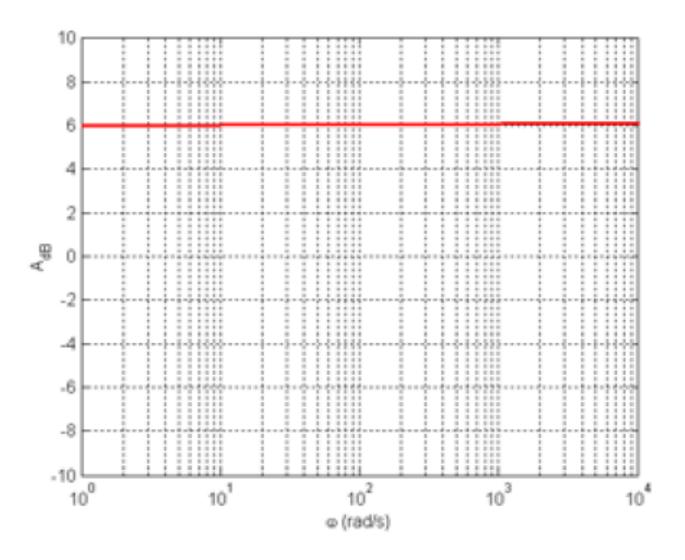
$$G = 2\frac{(1+0.1s)}{s(1+0.01s)}$$

$$A_{dB} = 20 \log \left( 2 \frac{1}{|j\omega|} \frac{|1+0.1j\omega|}{|1+0.01j\omega|} \right)$$

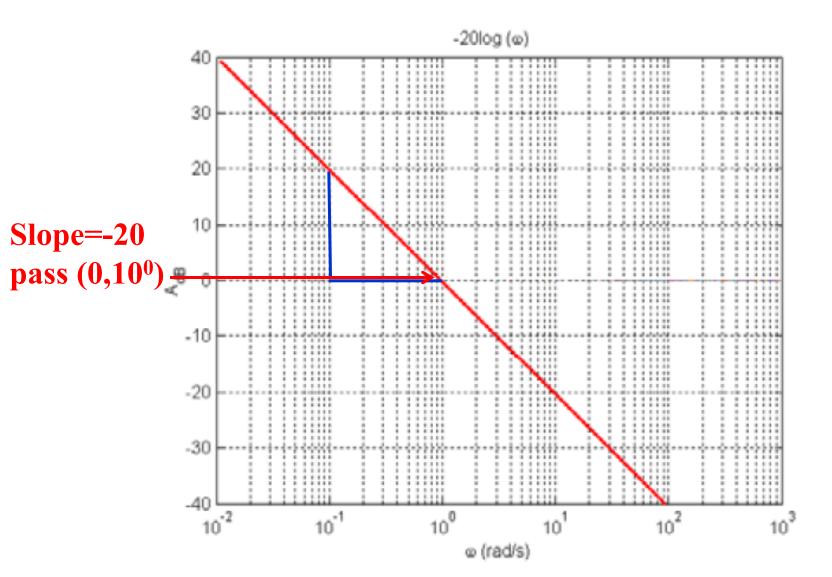
$$= 20\log(2) + 20\log|1 + 0.1j\omega|$$

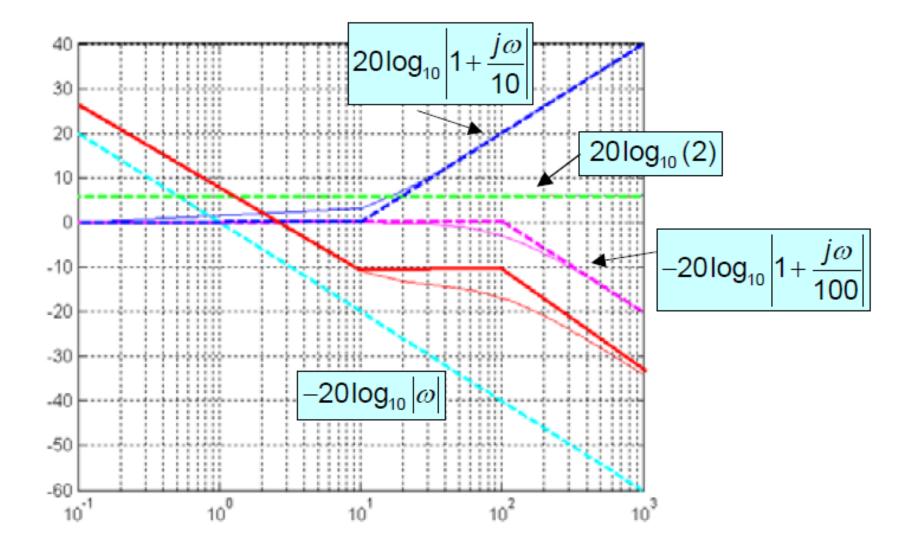
$$-20\log|j\omega| - 20\log|1 + 0.01j\omega|$$

$$A_{dB} = 20\log(2) = 6dB$$



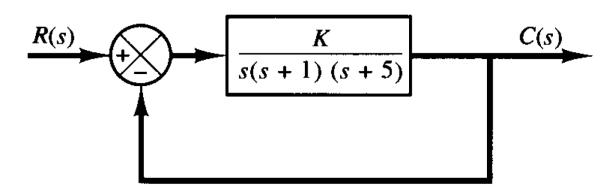
$$A_{dB} = -20\log|j\omega|$$





# Stability Analysis

• The phase and gain margins can easily be obtained from the Bode diagram.



• For K = 10 and K = 100, find the phase and gain margins

